

STS Propellant Densification Feasibility Study Data Book

M.M. Fazah
Marshall Space Flight Center • MSFC, Alabama

National Aeronautics and Space Administration
Marshall Space Flight Center • MSFC, Alabama 35812

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TECHNICAL MEMORANDUM

STS PROPELLANT DENSIFICATION FEASIBILITY STUDY DATA BOOK

I. INTRODUCTION

A. Need

The need for increased payload capability for launch vehicles is driven by ever changing requirements. The Space Transportation System (STS) or space shuttle, having the requirement to support the International Space Station, is examining options to increase its payload capability. Cryogenic propellant densification is a potential option for doing so.

Propellant densification is not a new approach. Slush hydrogen has been examined in the past for various programs. Operational considerations have prevented its use in the past. The use of subcooled propellant, above the slush point, may offer a solution. In order to assess the use of subcooled propellant on the shuttle or other vehicles, the system, technical, operational, requirements, and, most importantly, cost must be known. Propellant densification on the shuttle has been proposed by Rockwell International. This report is in response to that proposal, as well as the desire to maximize launch vehicle payload.

B. Objective

A product development team (PDT) has been formed to determine the feasibility of propellant densification in terms of technical, operational, and cost factors. The main objective of the PDT is to recommend a means of increasing the space shuttle payload by taking advantage of propellant densification, thus partially fulfilling the payload requirement for the International Space Station.

The reference vehicle that will be used as a baseline or bench mark is the current space shuttle. Various propellant densification concepts will be examined to determine their feasibility, as well as whether both liquid hydrogen (LH₂) and liquid oxygen (LO₂) should be densified or only one or the other. As the PDT product, a recommendation(s) will be made as to the most promising approach.

C. Goal

The goal of this report is to gain a thorough and in-depth understanding of propellant densification, to recommend a concept(s) for propellant densification, and to provide a performance, operational, and cost assessment for the concept(s). Figures 1 and 2 show the work breakdown schedule and the task logic diagram.

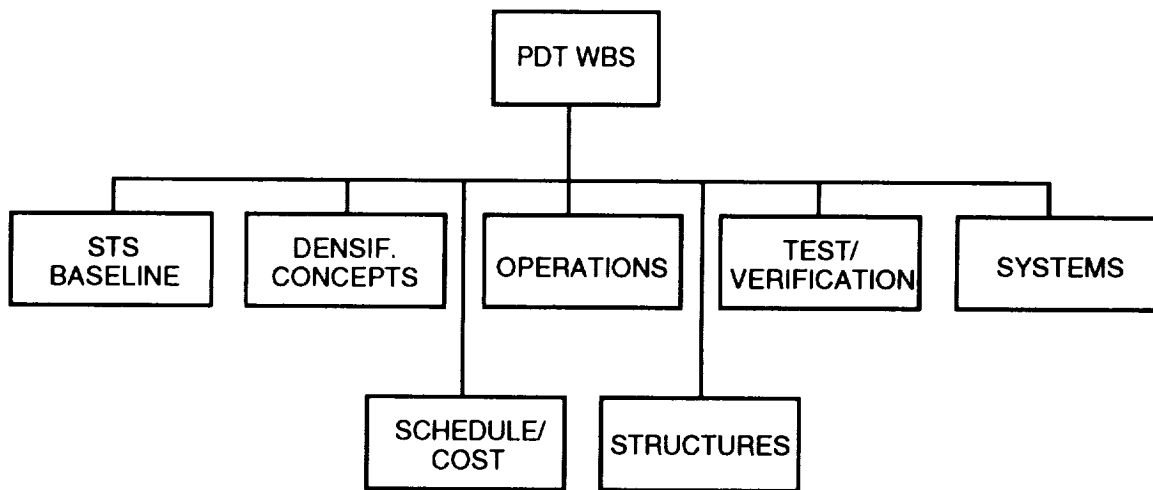


Figure 1. Work breakdown structure.

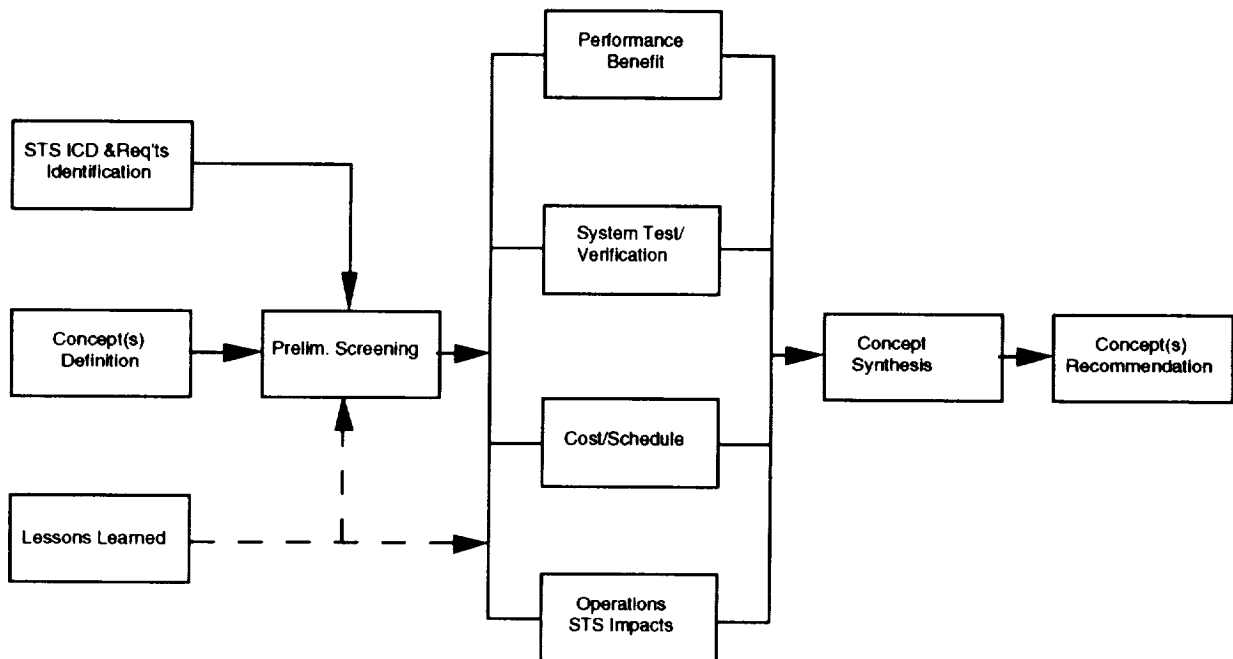


Figure 2. Task logic diagram.

II. ASSUMPTIONS/GROUND RULES

A. STS Enhancements

For the purposes of this study, the baseline vehicle will be the current STS, Johnson Space Center (JSC) Class 0, i.e., no enhancements.¹ Table 1 provides a summary of the ground rules. For further details, see reference 1.

Table 1. Reference trajectory.

Parameter	Value
Nominal Throttle Level	104 percent
Space Shuttle Main Engine (SSME) Rated Vacuum Thrust	470,259 lbf
SSME Rated Vacuum Isp	452.53 s
SSME Nominal Mixture Ratio	6.011:1
SRB PMBT	78 °F
Propellant Load at Startup	1,613,777 lbm
– LO ₂	1,382,028 lbm
– LH ₂	231,749 lbm
Main Engine Cut Off (MECO) Orbit Apogee	220 nm
MECO Orbit Perigee	31 nm
MECO Altitude	57 nm
Orbit Inclination	51.6°
Design Atmosphere	Summer
Design Winds	June
Maximum Dynamic Pressure (Q)	750 lb/ft ²
Minimum Q alpha	–3,000 lb/ft ² -°

III. STS BASELINE

A. External Tank (ET) Envelope

The densification subsystem shall operate within the existing dimensions of the ET as given in table 2.

Table 2. ET dimensions.

Description	LH ₂	LO ₂
Height (in)	1,160.75	588.4
Diameter (in)	330.0	330.0
Liq. Vol (ft ³) – Unpress	52,528	19,446
Liq. Vol (ft ³) – Press	53,152	19,672

B. ET Heat Flux

The nominal heat flux on the ET, for both the LH₂ and LO₂ tanks, are given in table 3.

Table 3. ET heat flux and vent rates.

Description	LH ₂	LO ₂
Q (Btu/s)	153.0	110.0
m _{vent} (lb/s)	0.77	1.2

C. Space Shuttle Main Engine (SSME) for Net Positive Static Pressure (NPSP)

The required NPSP for the low pressure LH₂ pump and the low pressure LO₂ pump are given in table 4.

Table 4. Required NPSP for SSME liquid propellant (LP) pumps.

Power Level	LH ₂ NPSP	LO ₂ NPSP
65	4.8	6.0
100	5.3	7.8
109	5.6	20.0

D. Existing Requirements

These current STS baseline requirements must be either met or revised if propellant densification systems are incorporated.

1. Launch Commit Criteria (LCC).

- LCC MPS-11 MPS LH₂ 17-Inch Manifold Disconnect Temperature/High-Point Bleed Temperature Anomaly
 - New maximum limits must be set for densified LH₂.
- LCC MPS-24 MPS LO₂ Engine Inlet Temperature Anomaly – High
 - LO₂ bleed flow temperature maximum limit will require change to reflect densified LO₂.
- LCC MPS-25 MPS LO₂ Engine Inlet Temperature Anomaly – Low
 - Lower temperature limit must be reestablished for densified LO₂.
- LCC MPS-33 Main Engine LH₂ Recirculation Anomaly
 - LH₂ recirculation interruptions (duration) will have to be reassessed for densified LH₂.
- LCC ET-04 ET LH₂ Prepress Cycle Anomaly
 - With colder LH₂, the number of prepress cycles required will likely increase.

2. Interface Control Document (ICD) Requirements.

- ICD-2-12001 Orbiter Vehicle/ET
 - Table 3.3.1-1 Orb/ET Fluid Separation Interface Conditions (Fluid Min, Nom, and Max Flowrate, Temperature and Pressure)
 - LH₂ Replenish
 - LH₂ Recirculation
 - LH₂ Drain (Detank)
 - LH₂ Engine Feed
 - GH₂ Pressurization
 - LO₂ Replenish
 - LO₂ Drainback
 - LO₂ Drain (Detank)
 - LO₂ Engine Feed
 - GO₂ Pressurization
 - Figure 3.3.2-1 Orb/ET Separation Interface LH₂ Prestart Requirements
 - Figure 3.3.2-2 Orb/ET Interface LH₂ Temperature Versus LH₂ Mass Remaining
 - Figure 3.3.2-4 Orb/ET I/F LH₂ Main Feed Line Propellant Requirements During Recirculation With LH₂ Tank Unpressurized
 - Figure 3.3.3-2 Orb/ET Interface LO₂ Main Engine Operation Requirements
- ICD-2-0A002 Space Shuttle Launch Pad and Platform
 - Table 4.3-1 ET Fluid Systems (Flowrate, Pressure and Temperature Limits)
 - GH₂ tank vent
 - GO₂ tank vent.

3. Operations and Maintenance Requirements and Specifications (OMRSD).

- LH₂ ullage pressure limits during various loading phases
- LO₂ ullage pressure limits during various loading phases.

E. New Requirements

Some new LCC, ICD, and OMRSD requirements would have to be established to incorporate densified propellants. Identifying new LCC and OMRSD requirements is beyond the scope of this study. However, it is likely that LCC's would be written to require a minimum amount of uninterrupted replenish time for each propellant to guarantee proper thermal conditioning.

- ICD-2-12001 Orbiter Vehicle/ET
 - Orb/ET interface conditions (fluid flowrate, temperature and pressure limits) for densified LH₂ recirculation line.
- ICD-2-0A002 Space Shuttle Launch Pad and Platform
 - Shuttle vehicle/pad interface conditions (fluid flowrate, temperature and pressure limits) for densified LH₂ recirculation line through LH₂ T-O umbilical and for LO₂ recirculation line through intertank.

IV. DENSIFICATION CONCEPT

A. Onboard Propellant Increase

STS payload gain can be achieved by increasing the effective density of cryogenic propellants, i.e., LH₂ and LO₂, by reducing the temperature of the liquid below that of the corresponding saturation temperature at ambient pressure. By densifying the propellant, i.e., increasing the effective density, more propellant mass can be loaded into the ET. This increase in propellant mass is translated into an increase in effective payload capability. Propellant densification can be accomplished on the LH₂ and LO₂ side either together or separately. Figures show the density versus temperature for LO₂ and LH₂.

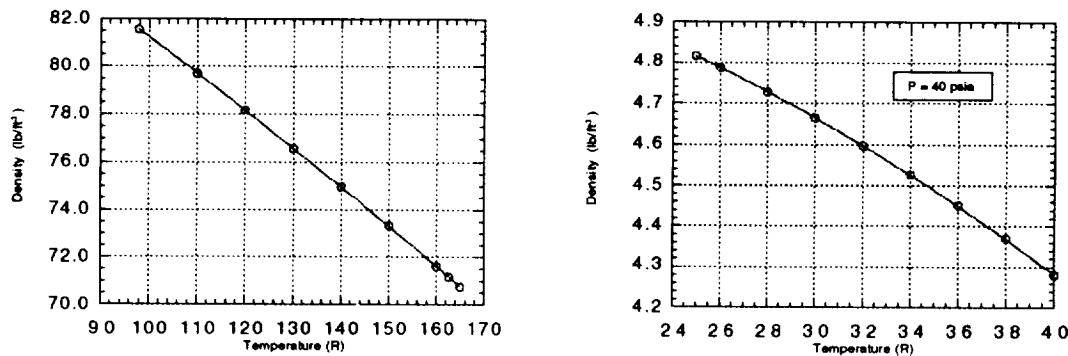


Figure 3. LO₂ and LH₂ density versus temperature.

Using the increased density propellant, an increase in the total propellant mass can be obtained. Table 5 and figure 4 show the propellant mass gain for LO₂ and LH₂.

1. LO₂/LH₂ Conditioning. The following analysis shows the propellant mass increase in the ET by densifying both the LH₂ and the LO₂. The current overboard mixture ratio is approximately 6.011. The propellant load can be calculated by keeping the overboard mixture ratio at 6.011, therefore, limiting the engine impact or maximizing the propellant mass onboard by going to an off nominal mixture ratio.

The LH₂ side is the driver, since the triple point is close to the nominal LH₂ temperature as compared to the LO₂ side. In section V, various concepts are examined to determine which is the most feasible for the LH₂ and LO₂ side. A minimum LH₂ temperature of 29 °R will be used in this analysis. This value has been determined by heat exchanger analysis performed by Rockwell International and will be used as the LH₂ temperature in all cases. For the LO₂ side, temperatures are in the range of 130 to 145 °R.

Two LO₂ temperatures, 140 and 130 °R, will be examined to assess the possible payload benefit. For both cases, the LH₂ temperature will be 29 °R. As given in appendix A, propellant inventories were performed on these cases to determine unusable residuals and the overboard mixture ratio required. These are summarized in table 6.

Table 5. LO₂ and LH₂ density versus temperature.

LH ₂ Parametrics				LO ₂ Parametrics			
Temp	Density	Mass Total	ΔMass	Temp	Density	Mass Total	ΔMass
24.987	4.8154	255,952	21,337	97.853	81.5632	1,604,505	206,245
25.00	4.8151	255,933	21,318	130.00	76.5819	1,506,514	108,253
25.50	4.8014	255,207	20,592	131.50	76.3413	1,501,780	103,520
26.00	4.7875	254,466	19,851	133.00	76.1000	1,497,032	98,772
26.50	4.7732	253,709	19,095	134.50	75.8579	1,492,270	94,010
27.00	4.7587	252,938	18,323	136.00	75.6150	1,487,493	89,233
27.50	4.7439	252,151	17,536	137.50	75.3715	1,482,702	84,442
28.00	4.7288	251,349	16,735	139.00	75.1272	1,477,897	79,636
28.50	4.7135	250,533	15,918	140.50	74.8822	1,473,077	74,816
29.00	4.6978	249,701	15,086	142.00	74.6365	1,468,243	69,982
29.50	4.6819	248,853	14,239	143.50	74.3900	1,463,394	65,134
30.00	4.6657	247,991	13,376	145.00	74.1428	1,458,531	60,271
30.50	4.6492	247,114	12,499	146.50	73.8949	1,453,654	55,393
31.00	4.6324	246,221	11,607	148.00	73.6462	1,448,762	50,502
31.50	4.6153	245,314	10,699	149.50	73.3968	1,443,856	45,595
32.00	4.5979	244,391	9,776	151.00	73.1467	1,438,935	40,675
32.50	4.5803	243,453	8,839	152.50	72.8958	1,434,001	35,740
33.00	4.5624	242,501	7,886	154.00	72.6442	1,429,051	30,791
33.50	4.5441	241,532	6,918	155.50	72.3919	1,424,088	25,827
34.00	4.5257	240,549	5,935	157.00	72.1389	1,419,110	20,849
34.50	4.5069	239,551	4,936	158.50	71.8851	1,414,117	15,857
35.00	4.4878	238,538	3,923	160.00	71.6306	1,409,111	10,850
35.50	4.4685	237,509	2,894	161.50	71.3753	1,404,090	5,829
36.00	4.4488	236,466	1,851	163.00	71.1193	1,399,054	794

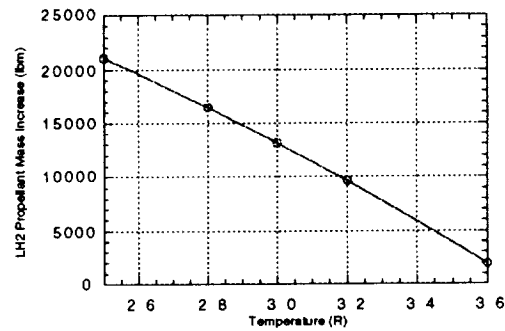
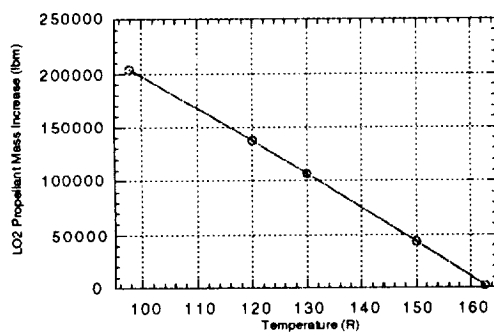


Figure 4. LO₂ and LH₂ propellant mass increase.

Table 6. Usable residuals and mixture ratios.

	LH ₂ Temp	LO ₂ Temp	LH ₂ Resid.	LO ₂ Resid.	M.R.
Option No. 1	28.5	141.5	2,050	4,887	5.93
Option No. 2	28.5	132.1	2,050	4,916	6.05

The delta propellant mass for these options are given in table 7.

Table 7. Delta propellant mass.

	LH ₂ Temp	LO ₂ Temp	Δ LH ₂	Δ LO ₂
Option No. 1	28.5	141.5	15,995	70,009
Option No. 2	28.5	132.1	16,002	100,004

2. **LH₂ Only Conditioning.** A LH₂ propellant mass gain can be accomplished by lowering the engine mixture ratio. As shown in section VII.A.2, the minimum mixture ratio that can be achieved is 5.61. Performing a propellant inventory for the LH₂ only case, for the minimum LH₂ temperature of 28.5 °R, the overboard mixture ratio is 5.64. This mixture ratio is within the minimum of 5.61.

Table 8 gives the delta propellant mass for the LH₂ only case.

Table 8. Delta propellant mass for LH₂ only case.

LH ₂ Temp	LH ₂ Resid.	Δ LH ₂	M.R.
28.5	2,050	15,997	5.64

B. Concept Definition—Trade Tree

Various concepts providing propellant densification will be studied to identify the concept or concepts that are the most appealing in terms of cost, performance, and operations. The most appealing option will have the least impact to the current STS system, while still offering a substantial payload gain.

Propellant densification can be provided either internal or external to the ET as shown in figure 5.

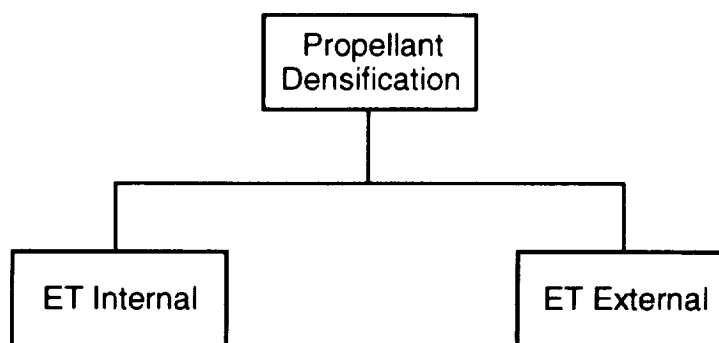


Figure 5. Propellant densification.

Depending on whether the densification is done internally or externally to the ET, various options are available. Also, densification may be performed on only the LH₂ or LO₂ side and may or may not be done on the propellant supply tanks.

1. ET Internal Densification. The trade tree given in figure 6 shows the various options for ET internal densification that will be considered in this study.

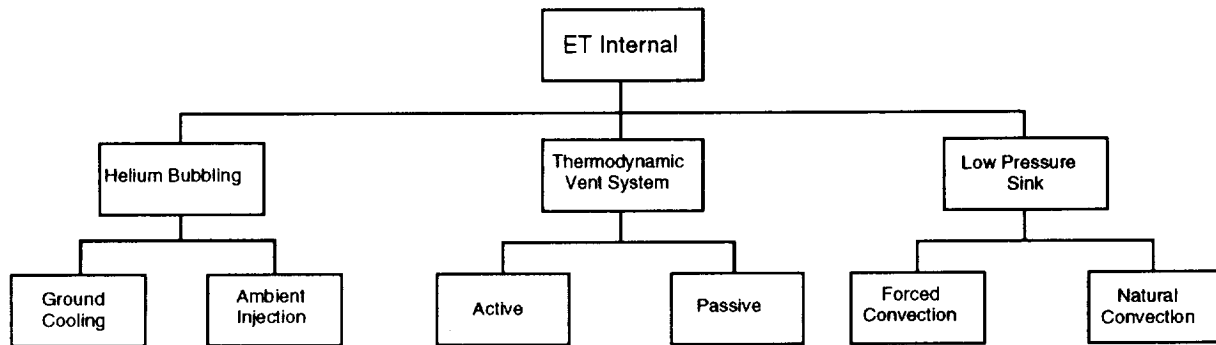


Figure 6. ET internal propellant densification.

(a) Helium (He) Bubbling. Propellant subcooling through He injection or He bubbling is obtained by evaporating LO_2 into the He bubbles and subsequently producing a cooling effect due to evaporation. This evaporation occurs due to the difference in LO_2 and He vapor pressure. As the LO_2 cools down and its vapor pressure decreases, the cooling effect becomes less, thus requiring large amounts of He to continue to cool the propellant. Due to LH_2 's molecular weight being lower than He, using He injection to cool LH_2 is not practical.

An additional concern with using He injection to cool the propellants is the loss of loading control due to the presence of a large number of He bubbles. The He injection must, therefore, be terminated prior to launch to gain liquid level control and then replenish with ambient temperature liquid as required. Also, the added concern of the He going into solution with the propellant and subsequently coming out may be a problem. If He coolers are necessary to prechill the He prior to injection, the concept simplicity will be compromised.

(b) Thermodynamic Vent System (TVS). Propellant cooling through the use of a TVS is accomplished by expanding a small portion of the fluid to be cooled or a separate working fluid to a lower pressure, thus lowering its temperature. This cooler fluid is then used to cool the bulk propellant by circulating it through a heat exchanger.

If the TVS is carried onboard the STS, i.e. inside the ET, the weight and cost associated with the system would not be practical.

(c) Low Pressure Sink. Propellant cooling through the use of a low pressure sink is accomplished by exposing the propellant to a pressure lower than ambient, thus saturating the propellant at a lower temperature. The desired temperature of the propellant can be controlled by the system back pressure.

By lowering the propellant vapor pressure, without prepressing the system, the tank/system positive pressure will be compromised, thus introducing tank structural and propellant contamination issues.

2. ET External Densification. The following trade tree (fig. 7) shows the various options for ET external densification that will be considered in this study.

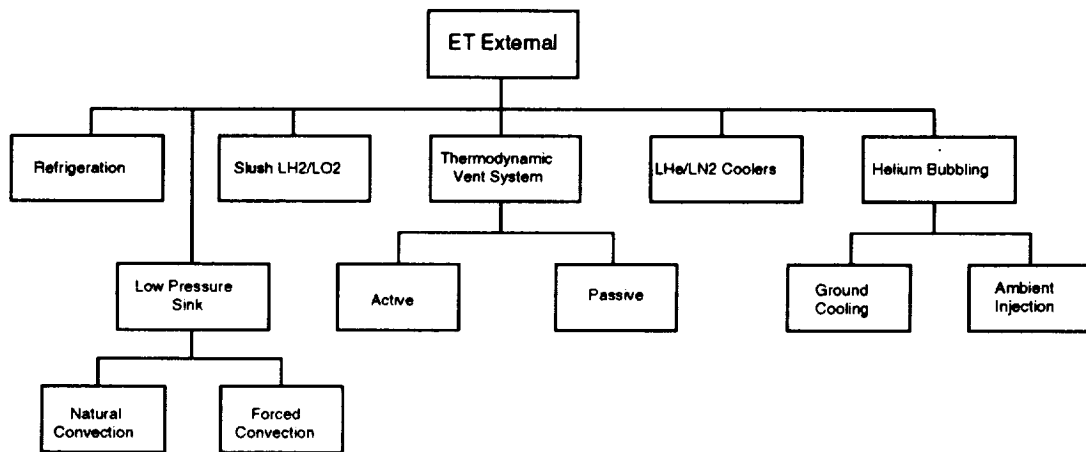


Figure 7. External propellant densification.

(a) Refrigeration. Propellant densification through refrigeration would require the use of complicated and expensive conditioning systems. An independent working fluid must be maintained to serve as the heat sink for the process. Components such as compressors, evaporators, and heat exchangers are required. Such a system becomes impractical for cryogenic systems.

(b) Low Pressure Sink. Propellant cooling through the use of a low-pressure sink is accomplished by exposing the propellant to a pressure lower than ambient, thus saturating the propellant at a lower temperature. The desired temperature of the propellant can be controlled by the system back pressure.

By lowering the propellant vapor pressure, without prepressing the system, the tank/system positive pressure will be compromised, thus introducing tank structural and propellant contamination issues.

(c) Slush LH₂/LO₂. Slush hydrogen or oxygen is the combination of the liquid and solid phases coexisting in solution. The temperature of the cryogen must be lowered to the triple point where solidification takes place. The advantage of slush is the large heat sink available due to latent heat from melting along with the latent heat from evaporation.

The disadvantages of slush hydrogen is that it must be mixed continually to prevent large solid hydrogen or oxygen particles from forming. Also, the cost of a slush production facility and the operational issues associated with its use have prevented slush hydrogen from being used in most vehicles under study in the past.

(d) Thermodynamic Vent System. Propellant cooling through the use of a TVS is accomplished by expanding a small portion of the fluid to be cooled or a separate working fluid to a lower pressure, thus lowering its temperature. This cooler fluid is then used to cool the bulk propellant by circulating it through a heat exchanger.

If the TVS is to be ground servicing equipment (GSE), the weight, operational impact, and recurring costs are significantly reduced.

(e) Liquid Helium (LHe)/Liquid Nitrogen (LN₂) Coolers. Propellant conditioning using coolers is accomplished by circulating the liquid propellant through a low-temperature bath, thus,

ideally, exiting at the bath temperature. This concept is relatively simple, since the propellant is only circulated through a passive bath—no active TVS or refrigeration systems are required.

The use of coolers is limited by the saturation temperature of the working fluid and by its heat of vaporization.

(f) **He Bubbling.** Propellant subcooling through He injection or He bubbling is obtained by evaporating LO₂ into the He bubbles and subsequently producing a cooling effect due to evaporation. This evaporation occurs due to the difference in LO₂ and He vapor pressure. As the LO₂ cools down and its vapor pressure decreases, the cooling effect becomes less, thus requiring large amounts of He to continue to cool the propellant. Due to LH₂'s lower molecular weight than He, using He injection to cool LH₂ is not practical.

An additional concern with using He injection to cool the propellants is the loss of loading control due to the presence of a large number of He bubbles. The He injection must, therefore, be terminated prior to launch to gain liquid level control and then replenish with ambient temperature liquid as required. Also, the added concern of the He going into solution with the propellant and subsequently coming out may be a problem. If He coolers are necessary to prechill the He prior to injection, the concept simplicity will be compromised.

V. PRELIMINARY CONCEPT SCREENING

The preliminary concept screening will entail weighting the concepts in four areas, conditioning performance, STS impacts, operational impacts and cost, in terms of high, medium or low merit; High being the most desirable and low being the least. The main goal of this section is to identify the concepts that are the most feasible for propellant densification. The top two concepts, on the LH₂ and LO₂ side, will be retained for more detailed analysis in subsequent sections.

A. Screening Criteria

The following categories will be used to screen the concepts identified in section IV.C.

- **Densification Performance (Perform):** A measure of the concepts ability to condition the propellant in 6 hours to some reference temperature based on its inherent thermodynamic and physical limitations.
- **Current System Impact (STS Impact):** A relative measure of the degree of impact required on the current system. This impact is measured in terms of physical changes and required testing. These are restricted to nonrecurring impacts.
- **Operational Impacts (Ops):** A relative measure of the degree of impact on the operations necessary to operate an STS configuration using this concept for propellant densification. These are restricted to recurring impacts.
- **Implementation Costs (Cost):** A relative measure of the cost associated with the specific concept. This includes all costs associated with the concept.

In tables 9 and 10, each concept is weighted High = 2, Medium = 1, and Low = 0 in each of the four areas. A score of 8 is the maximum and 0 is the minimum. For propellant conditioning inside the ET, He injection is the most viable in terms of the four areas described for the LO₂ side only. None of the options is viable for the LH₂ side.

Table 9. Internal propellant densification.

Concept	Perform.	STS Impact	Ops	Cost	Total Score
He Injection (LH ₂)					
- Ambient	0	2	0	1	3
- Cooled	0	2	0	1	3
Heat Exchanger (LH ₂)					
- Active	2	0	0	0	2
- Passive	1	0	0	0	1
Low Pressure Sink (LH ₂)					
- Natural Convection	2	0	0	1	3
- Forced Convection	2	0	0	0	2
He Injection (LO ₂)					
- Ambient	1	2	0	1	4
- Cooled	1	2	0	1	4
Heat Exchanger (LO ₂)					
- Active	2	0	0	0	2
- Passive	1	0	0	0	1
Low Pressure Sink (LO ₂)					
- Natural Convection	2	0	0	1	3
- Forced Convection	2	0	0	0	2

* High = 2, Medium = 1, Low = 0

Table 10. External propellant densification.

Concept	Perform.	STS Impact	Ops	Cost	Total Score
He Injection (LH ₂)					
- Ambient	0	2	0	1	3
- Cooled	0	2	0	1	3
Heat Exchanger (LH ₂)	2	2	1	1	6
Low Pressure Sink (LH ₂)					
- Natural Convection	2	0	0	1	3
- Forced Convection	2	0	0	0	2
Refrigeration (LH ₂)	2	2	1	1	6
Slush (LH ₂)	2	0	0	0	2
LHe Cooler (LH ₂)	2	2	1	1	6
He Injection (LO ₂)					
- Ambient	1	2	0	1	4
- Cooled	1	2	0	1	4
Heat Exchanger (LO ₂)	2	2	1	1	6
Low Pressure Sink (LO ₂)					
- Natural Convection	2	0	0	1	3
- Forced Convection	2	0	0	0	2
Refrigeration (LO ₂)	2	2	0	0	4
Slush (LO ₂)	2	0	0	0	2
LN ₂ Cooler (LO ₂)	2	2	1	1	6

* High = 2, Medium = 1, Low = 0

For propellant conditioning external to the ET, the following concepts are the most appealing:

- LH₂
 - LH₂ Heat exchanger
 - LHe Cooler
 - LHe/LH₂ Refrigerator
- LO₂
 - LN₂ heat exchanger
 - LN₂ Cooler

VI. CONCEPT REFINEMENT

A. Heat Exchanger Design Parameters/Feasibility

As shown in section V, the most feasible densification technique is to perform the conditioning external to the ET. The simplest heat exchanger design, which will give the required performance, is a constant temperature bath heat exchanger. For the LO₂ conditioning, a LN₂ can be used, and, for the LH₂ side, a subcooled LH₂ bath can be used. Figure 8 shows a typical constant temperature bath heat exchanger. The bath can be subcooled through the use of the compressor if lower bath temperatures are required.

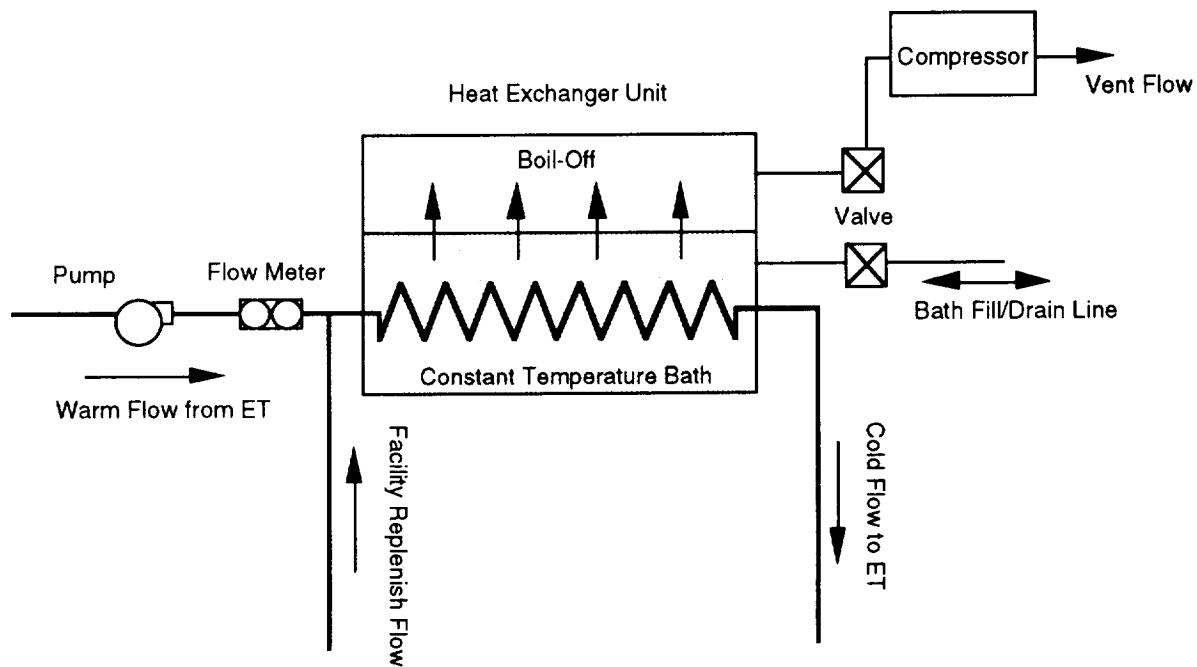


Figure 8. Typical constant temperature bath heat exchanger.

1. LO₂ Conditioning (LN₂ Bath). LN₂ can be used on the LO₂ side, since the saturation temperature of LN₂ is approximately 24 °R below the LO₂ saturation temperature at the same pressure. Table 11 gives saturation temperatures for various LN₂ pressures.

Table 11. LN₂ saturation temperature versus pressure.

Pressure (lb/in ² absolute)	Temperature (°R)
5.0	124.6
10.0	133.6
14.7	139.3

For option No. 1, no subcooling of the LN₂ is required; therefore, a LN₂ bath at ambient temperature, 139.3 °R and pressure, 14.7 lb/in² absolute can be used.

For option No. 2, the LN₂ subcooling required to achieve the 132 °R conditioning level is approximately 8 lb/in² absolute or a temperature of 130 °R.

2. LH₂ Conditioning (Subcooled LH₂ Bath). In order to condition the LH₂, a subcooled LH₂ bath will be required. Table 12 gives saturation temperatures for various LH₂ pressures.

Table 12. LH₂ saturation temperature versus pressure.

Pressure (lb/in ² absolute)	Temperature (°R)
1.0	25.0
5.0	30.8
10.0	34.3
14.7	36.5

For the LH₂ conditioning, the LH₂ bath must be subcooled to approximately 1.5 lb/in² absolute , or a temperature of 26 °R.

B. Conditioning Timeline Analysis

The time required to condition the propellant is critical in the development of a loading and pre-launch STS timeline. The following sections will examine the time required for both the LH₂ and LO₂ as a function of circulation flow rate and the conditioning system efficiency. Both tank conditions were modeled using an energy and mass balance on the system assuming complete mixing in the ET, as shown in figure 9.

An alternate approach to estimating the required mixing time is to assume no mixing at all, i.e., the warm fluid is pushed out of the tank or displaced by the cold fluid. This would demote the shortest conditioning time since only one change of tank mass would be required (fig. 10).

Based on the analysis in section VI.A, two LO₂ temperatures and one LH₂ temperature will be examined. These temperatures are shown in table 12.

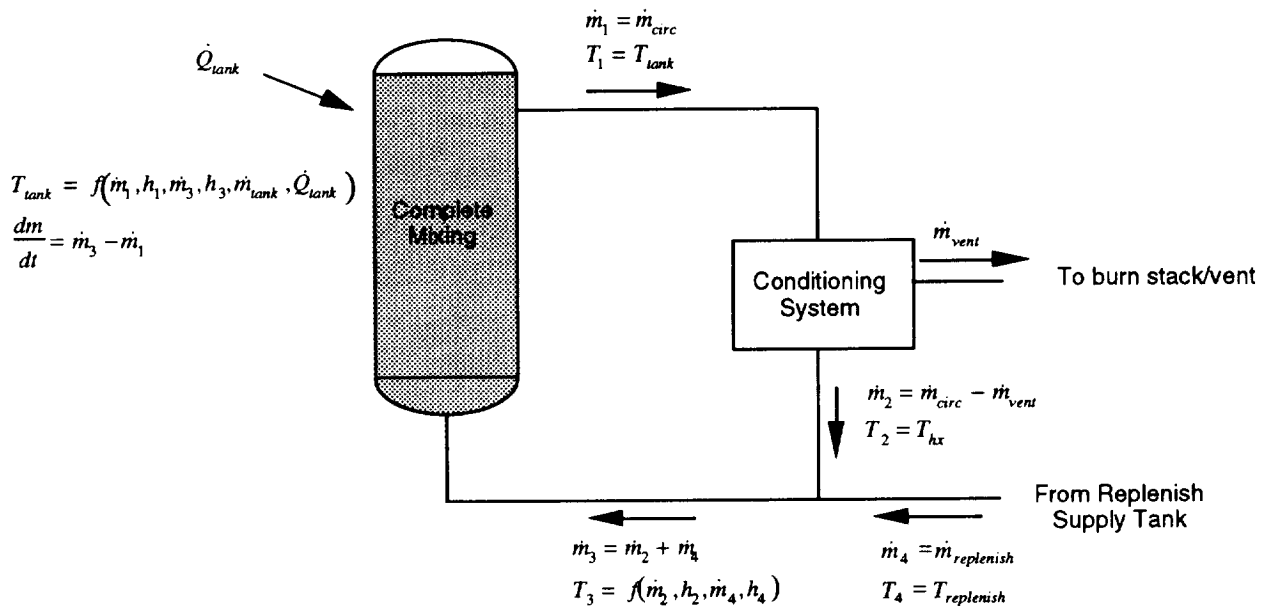


Figure 9. Thermodynamic analysis loop.

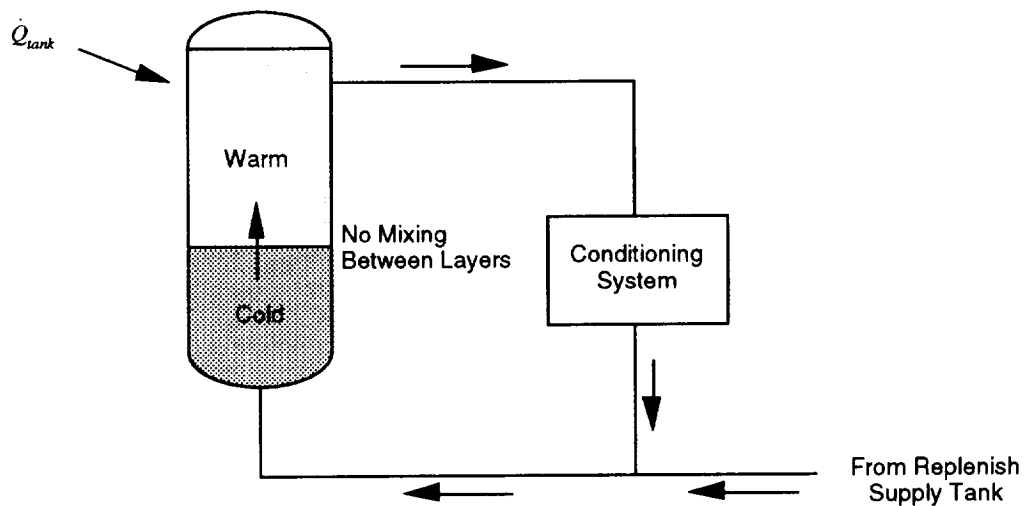


Figure 10. Propellant displacement analysis.

Table 13. LO₂ and LH₂ temperatures.

LO ₂ (°R)	LH ₂ (°R)
141.5	28.5
132.1	28.5

1. LN₂/LO₂ Cooler (LO₂ Temperature = 141.5 °R). The initial conditions for the LO₂ cooler conditioning analysis are given in table 14. Circulation flow rate was varied to determine the minimum flow rate that satisfies the densification performance criteria. This system is relatively inexpensive, since a LN₂ ambient bath will provide the temperature difference required. Therefore, no expensive or complicated vacuum systems are required.

Table 14. Typical LN₂/LO₂ cooler input file.

Fluid (1 = LH ₂ /2 = LO ₂)	= 2
Tank Liquid Volume	= 19,672.00 ft ³
Compressibility Factor	= 1.0
Tank Heat Flux	= 110.00 Btu/s
Initial Liquid Temp	= 164.00 °R
Supply Tank Temp	= 165.00 °R
Circulation Flow Rate	= as shown on plot
Heat Exchanger Exit Temp	= 140.00 °R
Simulation Time	= 30,000.00 s

The predicted temperature versus conditioning time is shown in figure 11.

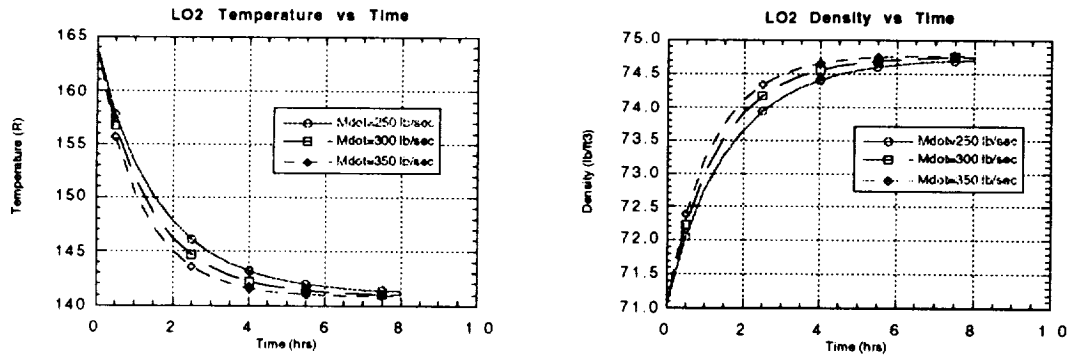


Figure 11. Predicted temperature versus conditioning time.

Based on figure 11, in order to meet the performance criteria of reaching the reference temperature of 141.5 °R in 6 h, a LO₂ flow rate of 300 lb/s will be used.

Based on the alternate analysis, i.e., colder fluid displacing the warmer fluid, the time required for the conditioning will be approximately 1.5 h. A detailed computational fluid dynamics (CFD) code and testing will be required to assess the extent of the mixing taking place. This leaves a range of conditioning time of 1.5 to 5.5 h for the LO₂, option No. 1.

The initial conditions for the LH₂ conditioning analysis are given in table 15. Circulation flow rate was varied to determine the minimum flow rate that satisfies the densification performance criteria. The LH₂ system is similar to the LO₂ system in that it uses a constant temperature bath to reject the heat. The major difference is that the bath must be subcooled LH₂, approximately 1.5 lb/in² absolute. This requires a compressor to subcool the LH₂.

Table 15. Typical LH₂ input file.

Ifluid (1 = LH ₂ /2 = LO ₂)	= 1
Tank Liquid Volume	= 53,152.00 ft ³
Compressibility Factor	= 0.995
Tank Heat Flux	= 154.00 Btu/s
Initial Liquid Temp	= 36.44 °R
Supply Tank Temp	= 38.00 °R
Circulation Flow Rate	= as shown on plot
Heat Exchanger Exit Temp	= 27.5 °R
Simulation Time	= 30,000.00 s

The predicted temperature versus conditioning time for various flow rates is shown in figure 12.

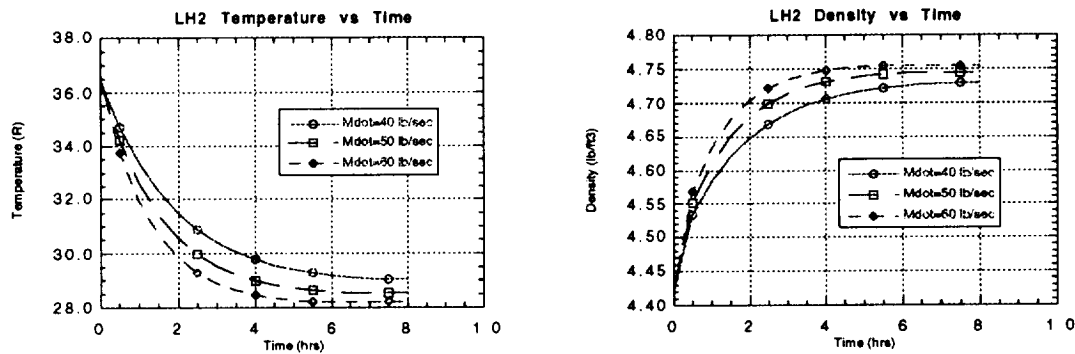


Figure 12. Predicted temperature versus conditioning time at various flow rates.

Based on figure 12, in order to meet the performance criteria of reaching the reference temperature of 28.5 °R in 6 h, a LH₂ flow rate of 50 lb/s will be used.

Based on the alternate analysis, i.e., colder fluid displacing the warm fluid, the time required for the conditioning will be approximately 1.5 h. A detailed CFD code and testing will be required to assess the extent of the mixing taking place. This leaves a range of conditioning time of 1.5 to 5.5 h for the LH₂.

2. LO₂/LN₂ Heat Exchanger (LO₂ Temperature = 132 °R). The initial conditions for the LO₂ heat exchanger conditioning analysis is given in table 16. Circulation flow rate was varied to determine the minimum flow rate that satisfies the densification performance criteria.

Table 16. Typical LN₂/LO₂ heat exchanger input file.

Ifluid (1 = LH ₂ /2 = LO ₂)	= 2
Tank Liquid Volume	= 19,672.00 ft ³
Compressibility Factor	= 1.0
Tank Heat Flux	= 110.00 Btu/s
Initial Liquid Temp	= 164.00 °R
Supply Tank Temp	= 165.00 °R
Circulation Flow Rate	= as shown on plot
Heat Exchanger Exit Temp	= 131.0 °R
Simulation Time	= 30,000.00 s

The predicted temperature versus conditioning time for various flow rates is shown in figure 13.

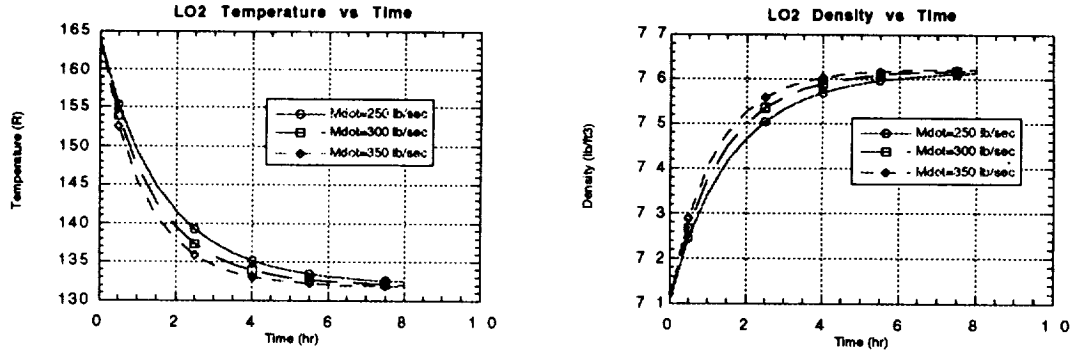


Figure 13. Predicted temperature versus conditioning time.

Based on figure 13, in order to meet the performance criteria of reaching the reference temperature of 130 °R in 6 h, a LO₂ flow rate of 350 lb/s (2210 gal/min at nominal conditions) will be used.

C. Propellant Temperature Uncertainty Analysis

Figure 14 shows the effect of the uncertainty in the liquid temperature of LO₂ and LH₂ on the uncertainty of the onboard propellant mass. As shown, large uncertainties in temperature produce smaller uncertainties in mass.

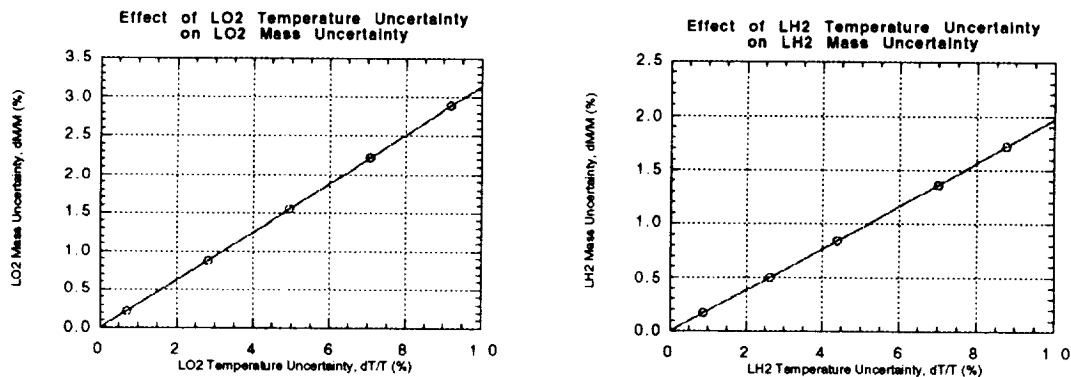


Figure 14. Effects of uncertainty.

D. Overall System Energy Balance

Given the preliminary nature of the overall designs, this study looked solely into the energy balance or heat transfer of the piping network necessary to supply the appropriate fluid to/from the heat exchanger. To perform this analysis, a System Improved Numerical Differencing Analyzer (SINDA) model was developed primarily from the analysis shown in section VI.B, with the addition of terms and equations to calculate heat transfer through the pipe network.

1. Assumptions. There were many assumptions made and boundary conditions had to be set in order to begin this analysis. To be consistent with previous analyses, the flow rates used for the LO₂ and

the LH₂ were set to 250 and 40 lbm/s and heat exchanger outlet temperatures (fluid outlet temperature) were set at 139 °R and 27 °R for LO₂ and LH₂, respectively. The approximate 600/200 ft of pipe was modeled with an inner diameter of about 8.5 in and outer diameter of about 10.25 in. The entire pipe network was modeled as a vacuum-jacketed system which consists of two pipes containing an evacuated space between them. The inner pipe was SS 304 with a multilayer insulation on its outer surface, while the surface of the outer SS 316L pipe was exposed to ambient outdoor conditions of 100 °F. Other parameters used were identical to those used in section VI.B.

2. Equations. The main emphasis in a vacuum-jacketed pipe is heat transfer via radiation. We assumed steady, uniform flow and a perfect vacuum space between the two pipes, thus eliminating any connective or conductive modes of heat transfer. It was first necessary to determine the effective emissivity of the whole pipe and to use that to determine the lost heat:

$$Q_{rad} = \sigma F_e A F (T_{in}^4 - T_{out}^4) ,$$

where

$$Q_{rad} = \text{heat due to radiation} ,$$

$$A = \text{surface area (ft}^2\text{)} ,$$

$$\sigma = \text{Stefan-Boltzman constant} = 0.1713\text{E-}8 \text{ Btu/ft}^2\text{-}^\circ\text{R}^4 ,$$

$$F = \text{Form Factor to space} = 1.0 ,$$

$$T_{in}^4 = \text{fluid or pipe temperature (}^\circ\text{R)} ,$$

$$T_{out}^4 = \text{outside temperature (}^\circ\text{R)} ,$$

$$F_e = \text{effective emissivity} ,$$

where

$$F_e = \left[\frac{1}{\epsilon_1} + \frac{A_1}{A_2} \left(\frac{1}{\epsilon_2} - 1 \right) \right]^{-1} ,$$

$$\frac{A_1}{A_2} = \text{surface area inside/surface area outside} ,$$

$$\epsilon_1 = \text{inside emissivity} = 0.04 ,$$

$$\epsilon_2 = \text{outside emissivity} = 0.85 .$$

These equations were placed in the model and added to energy balance at the tank. If one were to calculate a steady-state (nontime-dependent) heat loss due to radiation, the figures would be approximately 11,000 Btu/h for LO₂ and 4,000 Btu/h for LH₂. Compared to the 300,000 Btu/h plus loss through the tank insulation (see Mission Management Center (MMC) ET Thermal Data Book), these line losses are negligible. Figures 15 and 16 show the system used in this analysis.

The diagram illustrates the LO2 venting system architecture. It begins with the LO2 Tank, which connects to the LH2 Tank. The system includes an ET Interface and an FSS/MLP Interface. The LO2 TSM (Tank Safety Monitor) is shown with an 8" VJ (Vent Junction) and an 8" VJ (Vent Junction). The LO2 Vent Eng. Bleed is also indicated. The LO2 Disconnect Tower is a key component in the system. The LO2 Dump Basin is the final destination for the vented LO2. The LO2 Conditioner is shown in a detailed view, highlighting the internal flow paths and components like the RV, MEV, and various valves. The piping is labeled with diameters (e.g., 8" VJ, 6" VJ, 3" VJ) and elevations (e.g., EL 215', EL 195', EL 95').

Figure 15. KSC LO₂ system configuration.

PROPOSED LOCATION

The diagram illustrates the proposed location of the MLP Side 1 heat exchanger. It features a cross-section of the vehicle structure, including the LO₂ TANK, LH₂ TANK, and FIXED SERVICE STRUCTURE. A circular inset provides a detailed view of the heat exchanger location, showing connections from ET (External Tank) through valves (DV, AFV, MFV), filters (F), and a rockwell provided section to the heat exchanger (HX).

Legend:

- DV = DRAIN VALVE
- RV = REPLENISH VALVE
- MFV = MAIN FILL VALVE
- AFV = AUXILIARY FILL VALVE
- HX = HEAT EXCHANGER
- ET = EXTERNAL TANK
- F = FILTER

**MLP SIDE 1
APPROXIMATE LOCATION
HEAT EXCHANGER**

Figure 16. KSC LH₂ system configuration.

VII. STS SYSTEMS IMPACTS

A. Fluid System Integration

In order to adequately condition the propellants, they must be circulated into and out of the ET through an external conditioning system or heat exchanger to provide the required temperature drop. Figures 17 and 18 provide a representative fluid loop within the STS system.

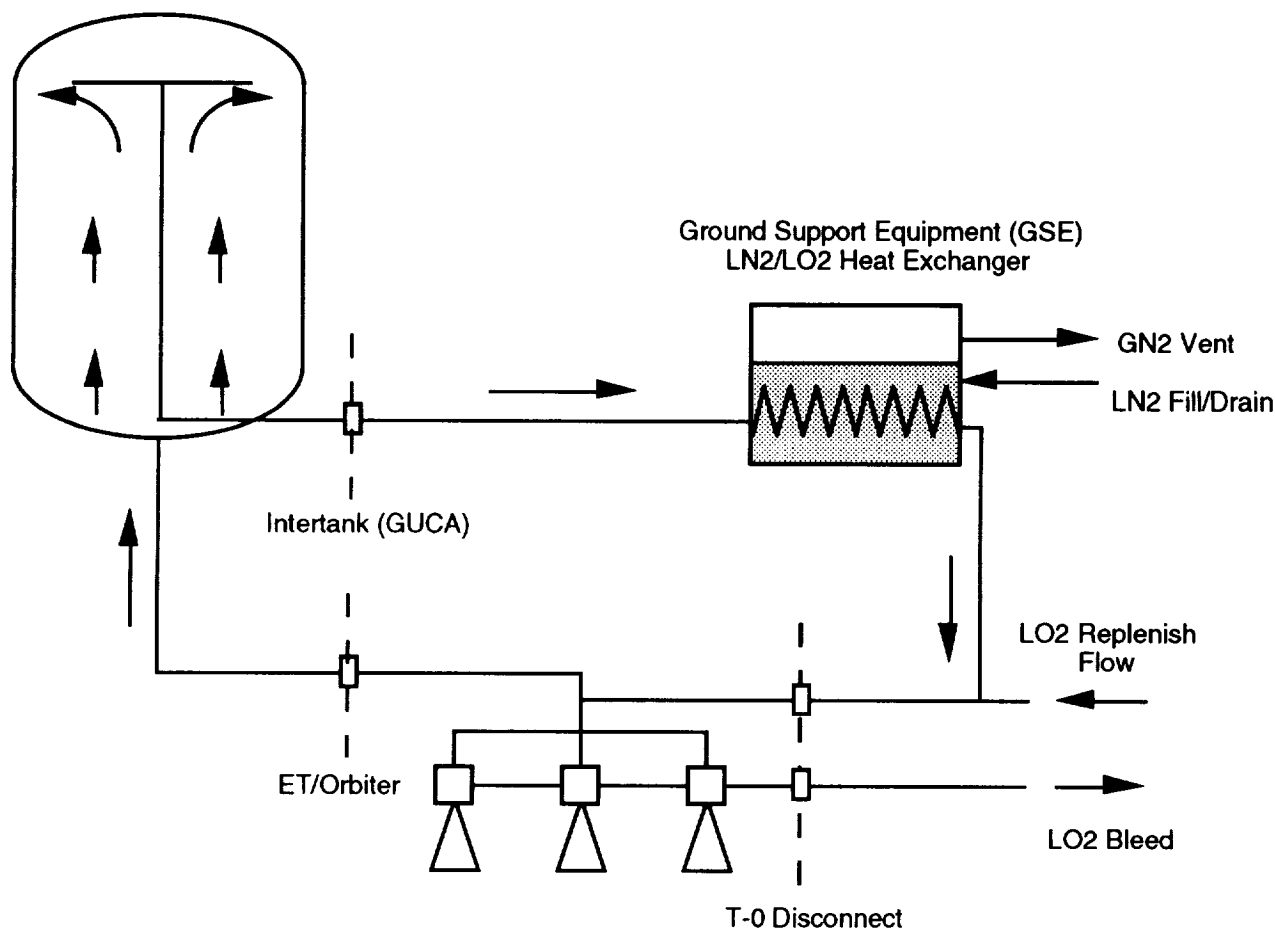


Figure 17. External conditioning system.

1. Description. As shown in figure 17, the conditioning system can be integrated within the current LO₂ fill system. The LO₂ will be circulated out of the tank, passed through a conditioning system to lower its temperature, and injected back into the tank. This loop will continue until the desired bulk temperature is reached. Internal to the ET will be a manifold system which will remove the fluid near the liquid surface and route it out of the tank through the intertank.

2. Hardware Modifications.

- (a) An ET LO₂ internal manifold system
- (b) LO₂ tank penetration near LO₂ feedline
- (c) Additional disconnect at the ground umbilical carrier assembly (GUCA)
- (d) Liquid level sensors for ± 100 percent.

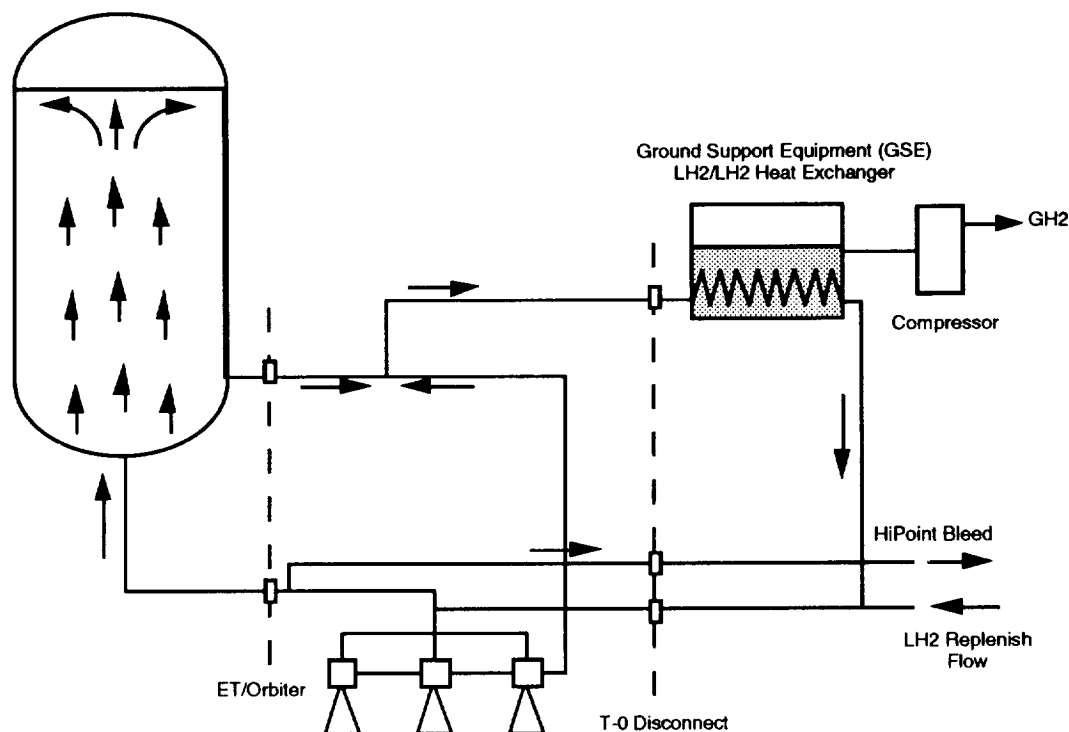


Figure 18. External conditioning system.

3. **Description.** As shown in figure 18, the conditioning system can be integrated within the current LH₂ fill system. As with the LO₂, the LH₂ will be circulated out of the tank, passed through a conditioning system to lower its temperature, and injected back into the tank. This loop will continue until the desired bulk temperature is reached. Internal to the ET will be a manifold system which will remove the fluid near the liquid surface and route it out of the tank through the intertank.

4. **Hardware Modifications.**

- (a) An ET LH₂ internal manifold system
- (b) Additional line within the orbiter boat tail
- (c) Additional disconnect at the T-0 plate
- (d) Liquid level sensors for ± 100 percent.

B. MPS Impact

Work on a steady-state model using denser LH₂ and LO₂ propellant to determine the minimum ullage pressure necessary to satisfy NPSP requirements for the SSME was performed. The model was built using data from Rockwell, the principles of basic fluids, and the EXCEL™ spreadsheet format to calculate the NPSP for each of the three engines of the shuttle.

Two options were considered for propellant densification: LO₂ temperatures at 141.5 and 132.1 °R. The LH₂ temperature remained constant at 28.5 °R for both LO₂ temperature options. The mixture ratio was 5.93 and 6.05, based on the temperatures set in options 1 and 2, respectively. Thus, in spite of the constant temperature of 28.5 °R for LH₂, two conditions would be generated for LH₂, based on the value of the mixture ratio. The guideline set for this model development was that the minimum NPSP for each of the shuttle engines would be no less than 6 lb/in² absolute for LH₂. The LH₂ value of 6 lb/in²

absolute is also being used for the current shuttle system. An ullage pressure was chosen arbitrarily to meet this minimum NPSP requirement. LO₂ actually has a minimum and maximum range of NPSP, which is currently being determined. However, for our model development, we again set the minimum NPSP at 6 lb/in² absolute. It was determined that flow control valves would be used for both LO₂ and LH₂. Therefore, a “press” model was deemed unnecessary for the LO₂ case. A “press” model would have been necessary if a fixed orifice application were used for LO₂. In that particular situation, results from the “press” model would be input into the spread sheet to assure that the NPSP requirements were met throughout the flight.

Results of this model development include four figures (figs. 19, 20, 21, and 22) of NPSP versus time generated for each of the conditions stated above. The general trends obtained from the figures are very similar to what is exhibited in NPSP versus time for the current shuttle system. This similarity in feedline characteristics shows that densifying the propellant should therefore be a feasible concept to apply in this respect, since feedline performance does not appear to be modified by the use of denser propellant. The EXCEL™ spreadsheet data which was used to compile the needed information and LH₂ and LO₂ tanking table data for use in the model development may all be found in the appendix. The EXCEL™ data also includes tank bottom pressures calculated for all of the above conditions.

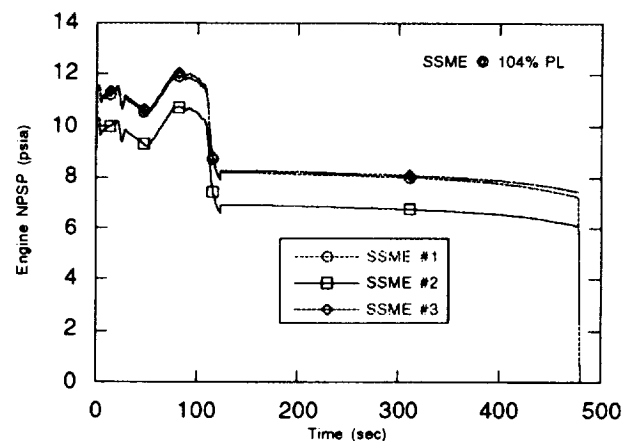


Figure 19. LH₂ NPSP for LO₂ at 132.2 °R.

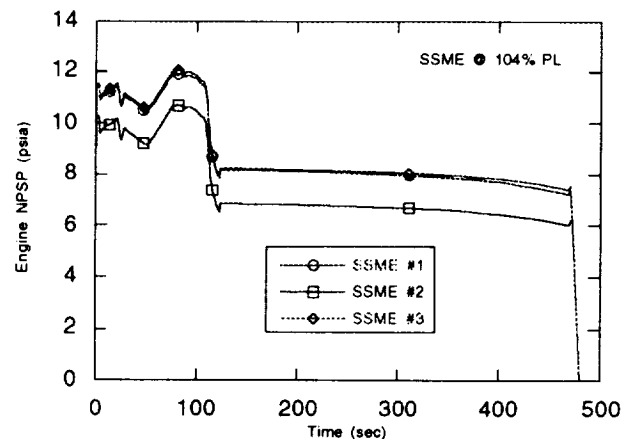


Figure 20. LH₂ NPSP for LO₂ at 141.5 °R.

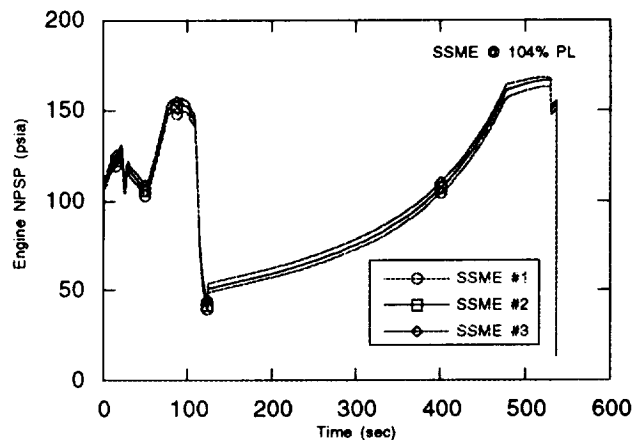


Figure 21. LO₂ NPSP at 132.1 °R.

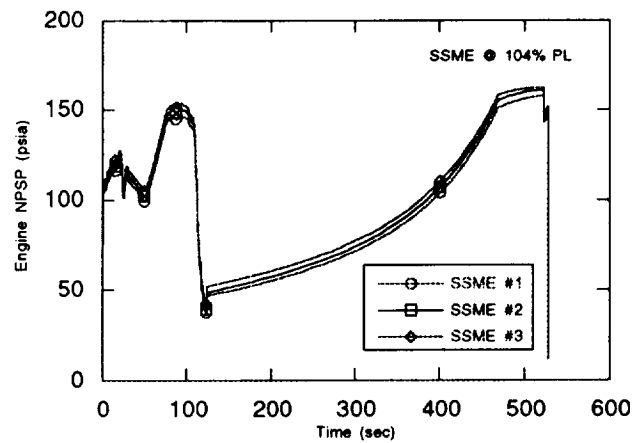


Figure 22. LO₂ NPSP at 141.5 °R.

C. Engine (SSME) Impact

The use of higher-density propellants will have an impact on the SSME start and operation due to their lower temperature. This impact will be assessed for LH₂/LO₂ densification and LH₂ only densification.

1. LH₂/LO₂ Conditioning. In order to maximize the total mass of propellant that may be added to the STS, operating at a higher mixture may be required. The following analysis gives the engine impact due to the higher mixture ratio shown in section IV.A.1.

2. LH₂ Only Conditioning. In order to obtain a propellant mass gain by conditioning the LH₂ only, the engine mixture ratio must be lowered to accommodate the increased LH₂ mass. Analysis was performed to identify the minimum mixture ratio that the engine can tolerate without violating high-pressure pump turbine temperatures or current redlines.

Table 17. SSME parameter sensitivity to mixture ratio (continued).

Current SSME (65 percent)				
<u>Engine Parameter</u>	<u>Baseline</u>	<u>dy/y per Percent MR</u>	<u>Del. Param.</u>	<u>Parameter</u>
Specific Impulse =	451.486 (s)	0.05	1.561	453.05
Thrust =	303,228 (lbf)	-0.13	-2,655.536	300,572.46
Mixture Ratio =	6.011	-1.00	-0.397	5.61
Fuel Flow Rate =	96.0239 (lbm/s)	0.65	4.148	100.17
Lox Flow Rate =	577.2 (lbm/s)	-0.33	-12.718	564.48
Total Flow Rate =	673.2239 (lbm/s)	-0.18	-8.144	665.08
HPFT Disch Temp =	1,484.85 (°R)	0.27	26.301	1,511.15
HPOT Disch Temp =	995.988 (°R)	-2.69	-177.280	818.71
FPOV Position =	68.034 (percent)	1.05	4.732	72.77
OPOV Position =	53.757 (percent)	-0.59	-2.090	51.67
LPFP Speed =	13,260.8 (r/min)	0.20	171.870	13,432.67
HPFP Speed =	27,029.4 (r/min)	0.27	490.450	27,519.85
HPFP Tot Ds Press =	3,926 (lb/in ² absolute)	0.35	91.591	4,017.59

Data Obtained from the Space Shuttle Main Engine Phase II Nominal Engine Power Balance & Operating Maximums Handbook, Jan. 31, 1990.

D. Tank Structural Impact

Due to the increased propellant mass, both LO₂ and LH₂, and the required circulation manifold system in the tanks, the super light weight tank (SLWT) or the current light weight tank (LWT) must be strengthened and will require additional hardware components to support propellant densification. Martin Marietta Corporation, the current ET contractor has assessed the impact of densification on the tank system. Figures 23 and 24 show the required manifold system routing and additional hardware required in the SLWT design.

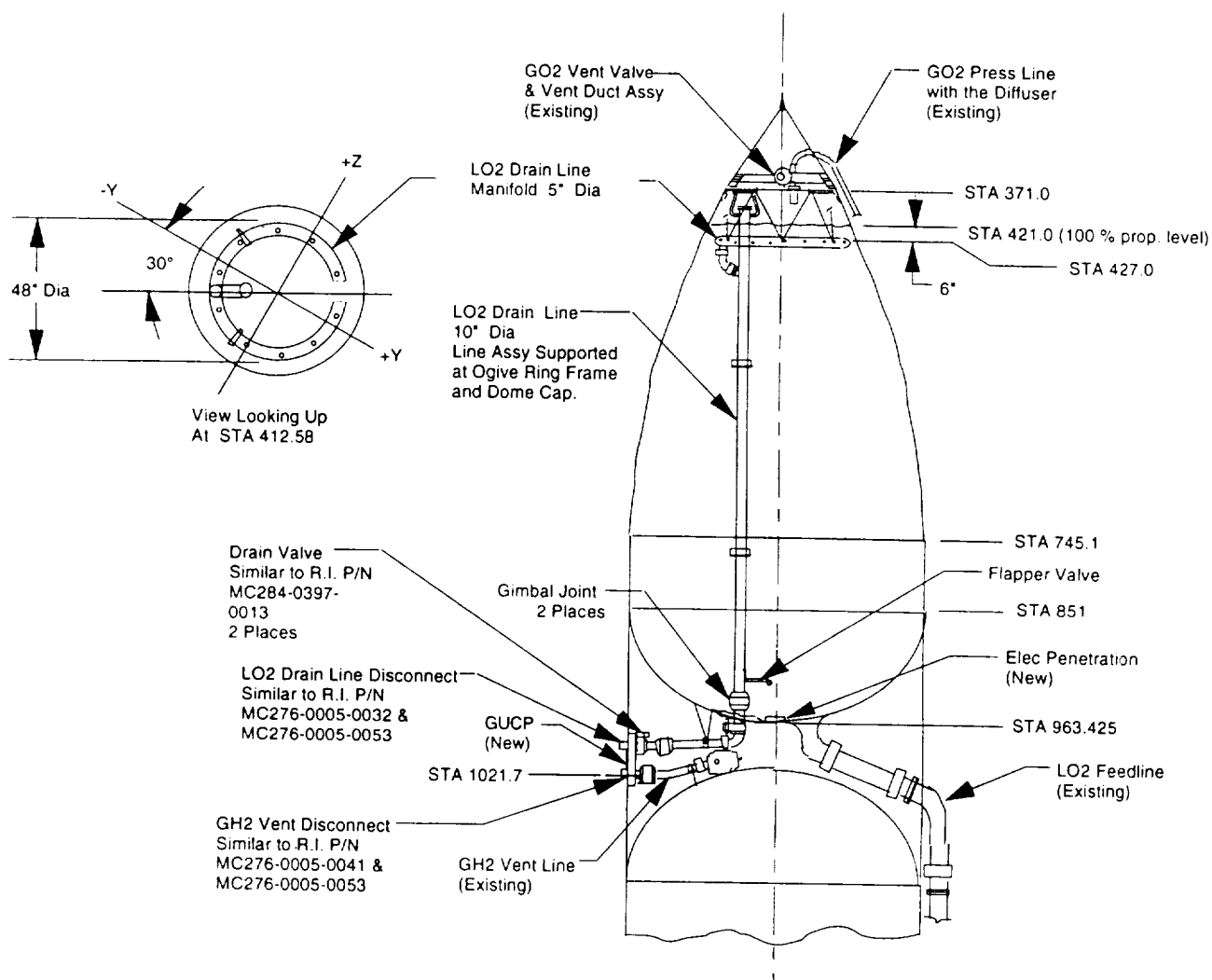


Figure 23. SLWT LO₂ tank design.

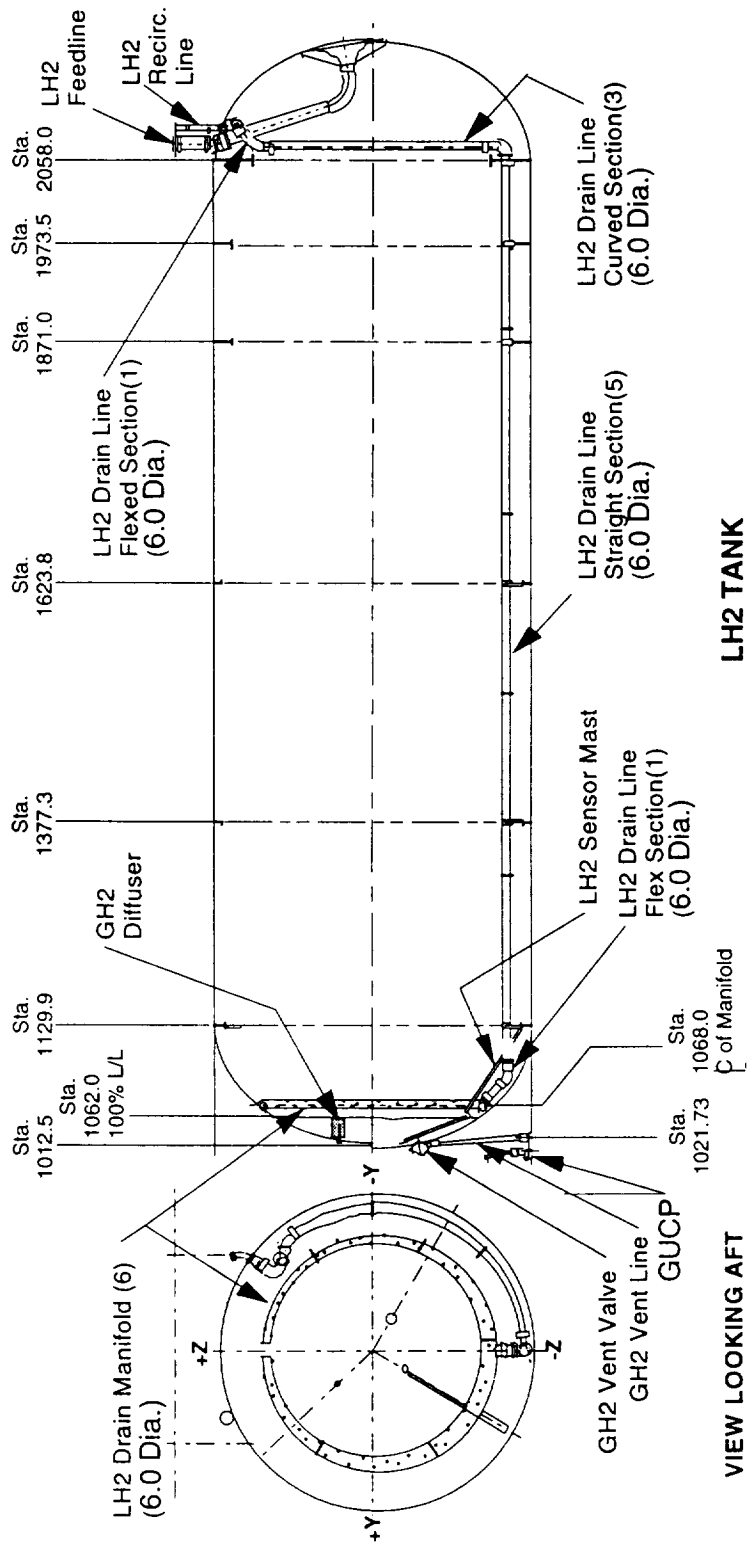


Figure 24. SLWT LH₂ tank design.

Table 18 summarizes the estimated weight and structural impacts to the SLWT due to densification.

Table 18. Major structural impacts.

Item	Est. Wt (lb)	Growth (lb)	Total (lb)
LO ₂ Structure			
• Aft Dome	77.0		
• Ogive	128.0		
	205.0	10.0	215.0
LH ₂ Structure			
• Barrel Sections	220.0	10.0	230.0
Intertank Structure			
• Thrust Panels	289.0		
• Skin/stringer Panels	86.0		
	375.0	19.0	394.0
Total Structural Impact			839.0

Table 19 summarizes the estimated weight of the LO₂ configuration, i.e., additional hardware required to support densification.

Table 19. LO₂ tank configuration.

Item	Qty	Est. Wt. (lb)	Growth (lb)	Total (lb)
Modify Sta 372 frame		0.0	0.0	0.0
Modify LO ₂ aft dome cap		0.0	0.0	0.0
Modify aft dome gore		20.0	3.0	23.0
New drain line supt.		20.0	3.0	23.0
New drain line bracket		10.0	1.5	11.5
New manifold brackets	12	30.0	4.5	34.5
Down comer line - 10 in. dia.	1	218.7	32.8	251.5
Toroidal manifold - 6 in	1	19.2	2.9	22.1
Flex line assy	1	26.6	0.0	26.6
Gimbal joint - 10 in	2	40.0	6.0	46.0
Drain valves	2	80.0	0.0	80.0
Drain line	1	44.0	0.0	44.0
Drain disconnect	1	15.0	0.0	15.0
Drain disconnect - non flight	1	0.0	0.0	0.0
TPS increase - aft dome		41.0	0.0	41.0
New GUCP - flight		30.4	0.0	30.4
Two stage vent valve		10.0	1.5	11.5
GUCP structure impact		10.0	1.5	11.5
Electrical hardware		8.2	1.2	9.4
Wire/sensors/feedthrough		65.3	9.8	75.1
Total Weight (lb)				756.0

Table 20 summarizes the estimated weight of the LH₂ configuration, i.e., additional hardware required to support densification.

Table 20. LH₂ tank configuration.

Item	Qty	Est. Wt. (lb)	Growth (lb)	Total (lb)
Fwd dome manifold mts	24	5.0	0.8	5.8
Mounting brackets		15.0	2.3	17.3
Frame mounting provisions		12.0	1.8	13.8
Sta 1871 mounting fitting		20.0	3.0	23.0
Sta 2058 hole and doublers	1	0.0	0.0	0.0
Upper drain line assy		21.0	0.0	21.0
Lower drain line assy		14.0	0.0	14.0
6-in drain manifold		65.6	9.8	75.4
Two stage vent valve	1	10.0	1.5	11.5
TPS increase-LH ₂ fwd dome		41.0	0.0	41.0
Electrical-sensors/wires		84.9	12.7	97.6
Drain reposition line		48.9	7.3	56.3
Down comer line - 6 in	1	153.7	23.1	176.7
Total Weight (lb)				553.0

The total weight impact due to densification is given in table 21.

Table 21. SLWT weight summary.

Item	Total Weight (lb)
LO ₂ Tank	971.0
LH ₂ Tank	783.0
Intertank	394.0
Total	2,148.0

VIII. PAYLOAD BENEFIT

To determine the performance improvement derived from propellant densification, trajectory analysis was done using MSFC shuttle models. The trajectories were flown using the new propellant inventories, structural and densification subsystem mass. To take full advantage of the changes in propellant loading, the engine mixture ratio was adjusted for each option. Therefore, the new runs also included changes to the SSME thrust and Isp due to the changes in mixture ratio. The trajectories were all flown according to the ground rules set forth in table 1. The performance improvement will be measured in the form of changes in orbiter weight at MECO.

The nominal MECO weight from which all delta payload values will be taken is given in table 22.

Table 22. Nominal payload.

Description	Total Prop	Δ Prop	MECO Wt.
Nominal Mission (104 percent throttle)	1,613,777	0	324,430
Nominal Mission (106 percent throttle)	1,613,777	0	324,817

A. LO_2/LH_2 Conditioning

Table 23 shows the performance gains due to conditioning both propellants. The LH_2 temperature for both options is 28.5 °R.

Table 23. LO_2 and LH_2 densification gross MECO weight gain.

Description	Total Prop	Δ Prop	MECO Wt.	Δ MECO Wt.
LO_2 at 141.5 °R (104 percent throttle)	1,699,781	86,004	330,521	6,091
LO_2 at 132.1 °R (104 percent throttle)	1,729,783	116,006	331,271	6,841
LO_2 at 141.5 °R (106 percent throttle)	1,699,781	86,004	331,322	6,505
LO_2 at 132.1 °R (106 percent throttle)	1,729,783	116,006	332,141	7,324

Figure 25 gives the total propellant increase versus gross payload gain, and table 24 gives the LO_2/LH_2 net payload gains.

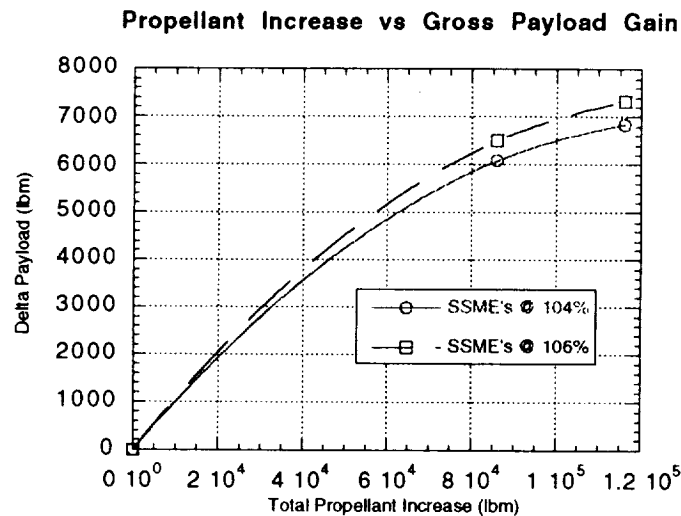


Figure 25. Total propellant increase versus gross payload gain.

Table 24. LO₂/LH₂ net payload gain.

Description	Total Prop	Δ Prop	Δ MECO Wt.
LO ₂ at 141.5 °R (104 percent throttle)	1,699,781	86,004	6,091
LO ₂ at 132.1 °R (104 percent throttle)	1,729,783	116,006	6,841
LO ₂ at 141.5 °R (106 percent throttle)	1,699,781	86,004	6,505
LO ₂ at 132.1 °R (106 percent throttle)	1,729,783	116,006	7,324

B. LH₂ Only Conditioning

Table 25 shows the performance gains derived from densifying only the LH₂ based on density values given in section IV.A and engine parameters given in section VII.C.

Table 25. LH₂ only densification gross MECO weight gain.

Description	Total Prop	Δ Prop	MECO Wt.	Δ MECO Wt.
LH ₂ Only at 28.5 °R (104 percent throttle)	1,629,774	15,997	325,618	1,188
LH ₂ Only at 28.5 °R (106 percent throttle)	1,629,774	15,997	326,120	1,303

IX. SYSTEM TEST/VERIFICATION

Test and verification requirements for the propellant densification systems and affected shuttle and the facility systems are presented in table 26. It should be noted that this test/verification plan does not require MPTA type testing. More detailed information on each item is presented in appendix B, System Test/Verification Detail Analysis.

X. TECHNICAL ISSUES

This section lists some of the technical issues that were raised during this study. This list is not considered complete and should be updated as other issues are raised.

(1) Density Assurance: In order to obtain orbit with the specified payload, the propellant mass must be predictable and known. To ensure adequate propellant mass, the liquid temperature must be known throughout the tank by using temperature sensors or the temperature must be predictable through analysis and/or test.

(2) Liquid Level Control: The existing data base of liquid level as a function of percent wet of the sensor is lost when cryogen is densified. An alternate technique for liquid level control must be developed, tested, and certified.

Table 26. STS propellant densification test/verification matrix.

Item No.	Item	Analysis	Bench Test	Subscale Test	Full Scale Test (1)	KSC Loading Test (2)	KSC FRF	Remarks
1.	GSE Component Performance	X	X					LO ₂ and LH ₂ densification system components testing.
2.	Propellant Recirculation Line Disconnects	X	X					Densified LO ₂ recirc. line disconnect thru intertank. Densified LH ₂ recirc. line disconnect thru LH ₂ T-0 umbilical.
3.	GSE Propellant Conditioning System Performance	X		X	X	X(P)		Systems testing of Item No. 1 components. Closed-loop control of recirc. flowrates, tank propellant temps, etc.
4.	Liquid Level Sensor Characterization	X	X	X				Liquid surface change due to subcooled propellant and near-zero boiloff. New percent wet algorithms and/or sensor arrangement.
5.	LH ₂ Pre-Pressurization Performance	X		X		X(S)		Increased ullage pressure decay due to subcooled LH ₂ .
6.	LO ₂ Pre-Pressurization Performance	X		X		X(S)		Increased ullage pressure decay due to subcooled LO ₂ .
7.	LH ₂ Flight Pressurization Performance	X		X				Ring manifold (heat sink) will be uncovered early in flight. Mainstage LH ₂ ullage pressure control band may be lowered with densified LH ₂ .
8.	LO ₂ Flight Pressurization Performance	X		X				LO ₂ ullage slump may be worse due to subcooling and uncovered ring manifold (heat sink). Will return to active GOX FCV system.
9.	KSC LH ₂ Procedures	X		X		X(P)		Loadings, stop flows, reverts, drains, contingency procedures, etc.
10.	KSC LO ₂ Procedures	X		X		X(P)		Loadings, stop flows, reverts, drains, contingency procedures, etc.
11.	LH ₂ Tank Stratification Characterization	X		X	X			Good mixing is required for temperature/density assurance.
12.	LO ₂ Tank Stratification Characterization	X		X	X			Good mixing is required for temperature/density assurance.
13.	LH ₂ Engine Recirculation Performance	X				X(S)		LH ₂ recirc. system performance should not be changed unless engine temp or hi-point bleedrate problems are indicated.
14.	LO ₂ Bleed Performance	X		X				Verify through analysis and single engine testing. Goal is to minimize bleed rate and thus drainback mass after repl. term.
15.	Orbiter LH ₂ Feed and Fill/Drain System Performance	X	X or ->	X				Analyze feed system pressure loss changes. Test valve performance with densified LH ₂ .
16.	Orbiter LO ₂ Feed and Fill/Drain System Performance	X	X or ->	X				Analyze feed system pressure loss changes. Test valve performance with densified LO ₂ .

Table 26. STS propellant densification test/verification matrix (continued).

17.	POGO System Performance	X				X		Engine mounted POGO accumulator. Verify performance through analysis and single engine testing.
18.	Nose cone & Intertank Purge Performance Verification	X					X(S)	Propellant densification recirc. lines and colder vent lines may require changes in GN ₂ purge gas pressures & temps.
19.	SSME Start/Shut Down Transient Performance	X						Engine start and shutdown at lower LO ₂ and LH ₂ temperatures.
20.	SSME Mainstage Operation	X			X			Engine operation at lower temperatures, higher densities and higher inlet pressures.
21.	LO ₂ Ullage Pressure Slump Verification	X			X			Ullage pressure slump is extremely important item to verify due to potential for loss of vehicle/crew if LO ₂ tank buckles.
22.	Systems Integration Verification	X					X	Propellant densification systems, return to active LO ₂ press control, change of LO ₂ and LH ₂ mainstage ullage press control bands, Block II SSME's, SLWT, Orbiter mods, cluster performance, twang verification, etc.

Notes:

(1) Stennis Space Center

(2) P = Primary test requirement, S = Secondary test requirement

(3) Holds/Reverts/Aborts: Liquid temperature control and/or liquid level control is required during extended holds, stop flows and reverts to prevent the venting of liquid. Propellant conditioning GSE must be able to maintain the liquid temperature at nominal levels during extended holds. For holds/stop flows after conditioning is terminated, the ability to reinitiate conditioning or drain back to an appropriate level is required.

(4) Dynamic Response Due to Conditioning Flow Rate: The MPS system, both existing and modified/added components, must be capable of withstanding dynamic regimes and environments due to the conditioning flow rate and densified liquid temperature. MPS system must be tested and certified at the densified conditions.

(5) LO₂ Ullage Slump: LO₂ ullage slump at lift off must be reassessed due to the colder liquid conditions and added heat sink due to the manifolding system required in the LO₂ tank.

(6) Loss of Propellant Conditioning: Ability to safe the system after GSE stop flow or failure during propellant densification operations.

(7) Loss of Tank Pressurization: During densification operations both LO₂ and LH₂ tank must be pressurized to ensure structural stability and to prevent contamination from entering the tank. Assurances must be provided to prevent loss of tank pressurization during densification or provide for safing procedures in case of failure.

XI. CONCLUSIONS

Propellant densification for the space shuttle or the STS is possible, and a payload gain can be realized. In this study, liquid subcooling of 28.5 °R is required for LH₂ and 132.1 and 141.5 °R for LO₂. The nominal liquid temperatures are 36.4 °R and 164 °R for the LH₂ and LO₂, respectively.

An important result of this study is that the most feasible method for conditioning the propellant was determined to be external to the tanks by using GSE located on the MLP. The LH₂ and LO₂ is circulated from the ET to heat exchangers located on the MLP which subcool the propellant prior to flowing back into the tanks. Both heat exchangers can consist of constant temperature baths. For the LH₂, a subcooled hydrogen bath is used, and for the LO₂, a nitrogen bath is used. These are considered the simplest heat exchangers that can be used. In this study, three cases were examined: (1) LH₂ at 28.5 °R and LO₂ at 132.1 °R, (2) LH₂ at 28.5 °R and LO₂ at 141.5 °R and (3) LH₂ at 28.5 °R only. For case (1), the nitrogen bath must be also subcooled below ambient to achieve the lower liquid temperature.

It was shown that for case (1) with the SSME's throttled to 104 percent, a payload increase of 6,841 lb can be obtained, and with the SSME's throttled to 106 percent, a payload increase of 7,324 lb can be obtained. It should be noted that throttling the SSME to 106 percent increases the flight risk to the engines.

It was shown that for case (2) with the SSME's throttled to 104 percent, a payload increase of 6,091 lb can be obtained, and with the SSME's throttled to 106 percent, a payload increase of 6,505 lb can be obtained.

It was shown that for case (3) with the SSME's throttled to 104 percent, a payload increase of 1,188 lb can be obtained, and with the SSME's throttled to 106 percent, a payload increase of 1,303 lb

can be obtained. For case (3), it was necessary to decrease the engine mixture ratio to 5.64 due to the extra hydrogen.

The payload increases give for cases (1), (2), and (3) take into account the added weight to the ET from the additional lines necessary for propellant recirculation and the increase in tank structure necessary due to the increase in total propellant load.

A detailed test requirements analysis was performed for the densified STS system, which identified specific tests and test areas that would be required for validation of propellant densification prior to first flight. The test requirements identified in section IX requires component and subsystem tests for the majority of the requirements and large scale testing for certain other subsystems. Although large scale testing is required, a flight ET is not required for use in the test program.

Although a cost estimate was not performed in this study, data obtained from a complimentary study lead by level II determined that the cost for implementation of propellant densification on the STS would run in excess of \$500 million. This high cost can be attributed to force fitting propellant densification into a system that was not designed to accommodate it. It should be noted that propellant densification may be viable for a new launch vehicle so that the operations can be designed into the system.

Due to the relatively low payload gain for the cost, propellant densification is not recommended for implementation on the shuttle.

REFERENCES

1. Generic Shuttle Lift Capability Trajectory Design Data Package, SPLCS028(02B), Jan. 27, 1994
2. NBS Cryogenic Propellant Tables
3. Space Shuttle Main Engine Phase II Nominal Engine Power Balance & Operating Maximums Handbook, Jan. 31, 1990.
4. Van Wylen and Sonntag, Fundamentals of Classical Thermodynamics, Third Edition
5. Crane, Flow of Fluids Through Valves, Fittings and Pipe, Technical Paper No. 410
6. Janna, William S., Introduction to Fluid Mechanics, Second Edition

APPENDIX A

STS Propellant Inventory

DENS-INV.XLS

MCRE, 02/07/94

MPS PROPELLANT INVENTORY STS-DENSE TANK

REV HHH

CNTR MR = 5.90223

INL MR = 5.917501

OBRM = 5.93042 I

FUEL BIAS = 976.0 I THROTTLE SETTING NOM/AOA = 104/104

TOTAL FPR = 3,987.68 I SIGMA = 3.0

DELTA FPR = 0

	LH ₂	LO ₂	TOTAL	
LOADED	247,873	1,457,909	1,705,782	
ORB LINES	266	3,471	3,737	I
SSME x 3	62	1,392	1,454	I
ET (HXT)=1,044.6, HUP=0.20, LXT=412.58, LUP=0.78)	247,544	1,453,046	1,700,590	I
LOAD PRIOR TO ENG START CMD DBT. 4M:55S	104	5,800	5,904	
BOILOFF, DRAINBACK, ETC.	104	5,800	5,904	I
LOAD ENGINE START COMMAND	247,769	1,452,109	1,699,878	
ORB LINES	266	3,471	3,737	
SSME x 3	62	1,392	1,454	
ET	247,440	1,447,246	1,694,686	
TRANSFERRED FROM ET TO SSME x3	62	181	243	I
LOSS FOR THRUST BUILDUP & SRB IGN DELAY	1,754	9,498	11,252	I
LOAD AT SRB IGNITION COMMAND	246,015	1,442,611	1,688,626	
ORB LINES	266	3,471	3,737	
SSME x 3	124	1,573	1,697	
ET	245,624	1,437,567	1,683,192	
UNUSABLE	2,050	4,887	6,937	
ORB LINES (LOX ECO T=0.398 NPSP=6.4)	266	513	779	I
SSME x 3	62	1,392	1,454	
ET WET WALLS, BELLOWS	0	175	175	I
ET: LH ₂ LINES & TANK; LOX LINES	720	0	720	I
FLIGHT PRESSURIZATION	1,002	2,807	3,809	I
USABLE RESERVES	1,551	3,412	4,964	
ORB LINES (FPR)	0	2,958	2,958	
SSME x 3	0	0	0	
ET (FPR)	575	454	1,030	
BIAS	976	0	976	
USABLE IMPULSE	242,413	1,434,312	1,676,725	
USED AT OBRM (5,93042)	241,603	1,432,808	1,674,411	
SHUTDOWN CONSUMPTION	748	1,323	2,071	
0 SSME's FROM NOM PCT THROTTLE SETTING	0	0	0	I
3 SSME's FROM 67 PCT THROTTLE SETTING	748	1,323	2,071	I
VENTED AFTER SSME VALVE CLOSURE	62	181	243	

Propellant Temperature, °R

28.38

141.69

Propellant Density, Vented Tank (Term Rpln)

4.71791

74.62180

Tank Volume, 100-Percent Level, Vented Tank (Term Rpln)

52,469.09

19,446.08

Propellant Density, Pressurized Tank (ESC)

Total Tank Volume, Pressurized Tank (ESC)

53,152.43

19,671.92

Delta Usable at OBRM (Ref. Rev. EE)

15,994.9

70,009.0

MPS PROPELLANT INVENTORY STS-DENSE TANK

REV HHH

NOM

CNTR MR = 6.02573

FUEL BIAS = 957.51 THROTTLE SETTING NOM/AOA = 104/104

INL MR = 6.041000

TOTAL FPR = 4,002.85

SIGMA = 3.0

OBMR = 6.0544

I

DELTA FPR = 0

	LH ₂	LO ₂	TOTAL	
<u>LOADED</u>	247,856	1,487,988	1,735,844	
ORB LINES	266	3,542	3,809	
SSME X 3	62	1,421	1,483	
ET (HXT)=1,044.6, HUP=0.20, LXT=412.58, LUP=0.78)	247,528	1,483,025	1,730,553	
<u>LOAD PRIOR TO ENG START CMD DBT. 4M:55S</u>	104	5,800	5,904	
BOILOFF, DRAINBACK, ETC.	104	5,800	5,904	I
<u>LOAD ENGINE START COMMAND</u>	247,752	1,482,188	1,729,940	
ORB LINES	266	3,542	3,737	
SSME x 3	62	1,421	1,454	
ET	247,424	1,447,225	1,724,649	
TRANSFERRED FROM ET TO SSME x3	62	184	246	
<u>LOSS FOR THRUST BUILDUP & SRB IGN DELAY</u>	1,754	9,498	11,252	I
<u>LOAD AT SRB IGNITION COMMAND</u>	245,998	1,472,690	1,718,688	
ORB LINES	266	3,542	3,809	
SSME x 3	124	1,605	1,729	
ET	245,608	1,467,543	1,713,151	
<u>UNUSABLE</u>	2,050	4,916	6,966	
ORB LINES (LOX ECO T=0.398 NPSP=6.4)	266	513	779	I
SSME x 3	62	1,421	1,483	
ET WET WALLS, BELLOWS	0	175	175	I
ET: LH ₂ LINES & TANK; LOX LINES	720	0	720	I
FLIGHT PRESSURIZATION	1,002	2,807	3,809	I
<u>USABLE RESERVES</u>	1,525	3,435	4,960	
ORB LINES (FPR)	0	3,029	3,029	
SSME x 3	0	0	0	
ET (FPR)	575	406	973	
BIAS	958	0	958	
<u>USABLE IMPULSE</u>	242,423	1,464,339	1,706,762	
USED AT OBMR (5,93042)	241,613	1,462,832	1,704,445	
SHUTDOWN CONSUMPTION	748	1,323	2,071	
0 SSME's FROM NOM PCT THROTTLE SETTING	0	0	0	I
3 SSME's FROM 67 PCT THROTTLE SETTING	748	1,323	2,071	I
VENTED AFTER SSME VALVE CLOSURE	62	184	246	

Propellant Temperature, 'R	28.39	132.13	I
Propellant Density, Vented Tank (Term Rpln)	4.71760	76.26344	
Tank Volume, 100-Percent Level, Vented Tank (Term Rpln)	52,469.08	19,446.08	I
Propellant Density, Pressurized Tank (ESC)			
Total Tank Volume, Pressurized Tank (ESC)	53,152.43	19,671.92	I
Delta Usable at OBMR (Ref. Rev. EE)	16,002.2	10,004.5	

(LO2 DELTA IS NONE ZERO DUE TO CHANGE IN FB & FPR)

MPS PROPELLANT INVENTORY STS-DENSE TANK				REV HHH
NOM				CNTR MR = 5.61373
FUEL BIAS = 978.4 I THROTTLE SETTING NOM/AOA = 104/104				INL MR = 5.629000
TOTAL FPR = 3,985.71 SIGMA = 3.0				OBRM = 5.64073 I
DELTA FPR = 0				
	LH ₂	LO ₂	TOTAL	
<u>LOADED</u>	247,905	1,387,844	1,635,749	
ORB LINES	266	3,304	3,570	
SSME x 3	62	1,325	1,387	
ET (HXT)=1,044.6, HUP=0.20, LXT=412.58, LUP=0.78)	247,577	1,383,215	1,630,792	
<u>LOAD PRIOR TO ENG START CMD DBT. 4M:55S</u>	104	5,800	5,904	
BOILOFF, DRAINBACK, ETC.	104	5,800	5,904	I
<u>LOAD ENGINE START COMMAND</u>	247,801	1,382,044	1,629,845	
ORB LINES	266	3,304	3,570	
SSME x 3	62	1,325	1,387	
ET	247,473	1,377,415	1,624,888	
TRANSFERRED FROM ET TO SSME x3	62	172	234	
<u>LOSS FOR THRUST BUILDUP & SRB IGN DELAY</u>	1,754	9,498	11,252	I
<u>LOAD AT SRB IGNITION COMMAND</u>	246,047	1,372,546	1,618,593	
ORB LINES	266	3,304	3,570	
SSME x 3	124	1,497	1,621	
ET	245,657	1,367,745	1,613,402	
<u>UNUSABLE</u>	2,050	4,820	6,870	
ORB LINES (LOX ECO T=0.398 NPSP=6.4)	266	513	779	I
SSME x 3	62	1,325	1,387	
ET WET WALLS, BELLOWS	0	175	175	I
ET: LH ₂ LINES & TANK; LOX LINES	720	0	720	I
FLIGHT PRESSURIZATION	1,002	2,807	3,809	I
<u>USABLE RESERVES</u>	1,579	3,386	4,964	
ORB LINES (FPR)	0	2,791	2,791	
SSME x 3	0	0	0	
ET (FPR)	600	594	1,195	
BIAS	978	0	978	
<u>USABLE IMPULSE</u>	242,418	1,364,340	1,606,759	
USED AT OBRM (5,93042)	241,608	1,362,845	1,604,454	
SHUTDOWN CONSUMPTION	748	1,323	2,071	
0 SSME's FROM NOM PCT THROTTLE SETTING	0	0	0	I
3 SSME's FROM 67 PCT THROTTLE SETTING	748	1,323	2,071	I
VENTED AFTER SSME VALVE CLOSURE	62	172	234	

Propellant Temperature, 'R	28.36	163.10	I
Propellant Density, Vented Tank (Term Rpln)	4.71853	71.13078	
Tank Volume, 100-Percent Level, Vented Tank (Term Rpln)	52,469.08	19,446.09	I
Propellant Density, Pressurized Tank (ESC)			
Total Tank Volume, Pressurized Tank (ESC)	53,152.43	19,671.92	I
Delta Usable at OBRM (Ref. Rev. EE)	15,997.1	18.4	

(LO2 DELTA IS NONE ZERO DUE TO CHANGE IN FP & FPR)

APPENDIX B

System Test/Verification Detail Analysis

STS Propellant Densification Test Requirement Analysis

Item No.: 1 **Requirement:** GSE Component Performance

Description: Component level characterization of the LH₂ and LO₂ GSE heat exchanger, circulation pumps, compressors and valves. Heat exchanger performance characterization/verification in terms of energy removal rates, efficiency and performance throughout the densification range and timeline.

Analytical Models: Analytical models shall be developed by the selected contractor and/or component vendors

Test Requirements: Testing is required on heat exchanger to determine heat transfer surface area capacity/effectiveness, compressor(s) performance with one and two compressors in operation, LH₂ and LO₂ pump and valve operation.

Risk Assessment: Could jeopardize schedule, cost and performance of subsequent densification system testing (see Item 3).

STS Propellant Densification Test Requirement Analysis

Item No.: 2 **Requirement:** Propellant Recirculation Line Disconnects

Description: Design verification of densified propellant recirculation line disconnects. Preliminary locations of disconnects are through the ET Intertank for LO₂ and the LH₂ T-0 umbilical for LH₂.

Analytical Models: None required

Test Requirements: Test for fit/alignment, leaks, cold flow and separation dynamics. Utilize KSC Launch Equipment Test Facility (LETF).

Risk Assessment: Failure to operate as designed could result in Crit 1 failure from leakage during propellant densification or leakage/rupture during liftoff.

STS Propellant Densification Test Requirement Analysis

Item No.: 3 **Requirement:** GSE Propellant Conditioning System Performance

Description: System level characterization of the LH₂ and LO₂ GSE conditioning equipment. Chill down, start, operation, shut down and purge operations. Evaluate operation envelop and contingencies for stop flow conditions, reverts, etc.

Analytical Models: System level analytical models shall be developed by the selected contractor.

Test Requirements: System level test on subscale test article as a minimum. Required evaluation/verification of predicted conditioning times and levels. Evaluation and verification of operation coupled with liquid level and temperature control of the LH₂ and LO₂ tanks. Complete subscale verification required prior to KSC propellant loading tests.

Risk Assessment: Inadequate propellant densification system performance during nominal and contingency operations could result in scrubbed launches or launching with insufficient propellant margin.

STS Propellant Densification Test Requirement Analysis

Item No.: 4 **Requirement:** Liquid level sensor characterization

Description: Determine percent wet characterization of level sensors under densified propellant conditions. Design level sensor clustering to provide good liquid level determination during topping and replenish.

Analytical Models: Develop algorithms for level sensor percent wet for subcooled propellant surface. Also, Martin Marietta Nastran model that determines External Tank volumes versus tank height for following cases: (1) ambient vented, (2) ambient pressurized, (3) cryo-loaded, vented and (4) cryo-loaded, pressurized.

Test Requirements: Subscale tank testing using densified LO₂ and LH₂ to determine sensor responses to being cover by propellant surface with little or no boiling. If % wet algorithm that is derived from tests is too "flat", then determine a new ET sensor arrangement that would provide good level determination for precise topping and replenish control.

Risk Assessment: Lack of precise liquid level control during replenish would increase the uncertainty in propellant load and require an increase in the Flight Performance Reserve (FPR) allotted for loading accuracy.

STS Propellant Densification Test Requirement Analysis

Item No.: 5 **Requirement:** LH₂ Pre-Pressurization Performance

Description: Due to densified LH₂ load with subcooled surface, the ullage pressure decay rate during pre-press will increase. Determine analytically the amount of He (i.e., the number of bursts) required to stabilize the ullage pressure within the required pre-ignition band. Verify with subscale tank test and then with KSC loading test.

Analytical Models: Martin Marietta Single-Node Pressurization Model.

Test Requirements: First, establish pre-press control band required by engines and tank structure. Test initially at SSC on subscale tank with densified LH₂. Pre-press with GHe and determine number of one second bursts required to stabilize ullage pressure. Run several tests to determine variability. Repeat for KSC loading tests to establish final pre-press characterization and max number of He bursts above which a press system or tank leak is indicated.

Risk Assessment: Poor pre-press performance can cause ullage pressure to fall outside of required engine ignition band. Poor characterization of pre-press can cause tank or press system leaks to go unnoticed.

STS Propellant Densification Test Requirement Analysis

Item No.: 6 **Requirement:** LO₂ Pre-Pressurization Performance

Description: Due to densified LO₂ load with subcooled surface, the ullage pressure decay rate during pre-press will increase. Determine analytically the amount of He (i.e., the number of bursts) required to stabilize the ullage pressure within the required pre-ignition band. Verify with subscale tank test and then with KSC loading test.

Analytical Models: Martin Marietta Single-Node Pressurization Model.

Test Requirements: First, establish pre-press control band required by engines and tank structure. Test initially at SSC on subscale tank with densified LO₂. Pre-press with GHe and determine number of one second bursts required to stabilize ullage pressure. Run several tests to determine variability. Repeat for KSC loading tests to establish final pre-press characterization and max number of He bursts above which a press system or tank leak is indicated.

Risk Assessment: Poor pre-press performance can cause ullage pressure to fall outside of required engine ignition band. Poor characterization of pre-press can cause tank or press system leaks to go unnoticed.

STS Propellant Densification Test Requirement Analysis

Item No.: 7 **Requirement:** LH₂ Flight Pressurization Performance

Description: Two changes may affect LH₂ tank pressurization system performance. (1) A lowering of the mainstage pressure below the current 32-34 psia. This may be implemented along with densified LH₂. (2) The presence of the LH₂ recirculation manifold will likely act as a heat sink when it is uncovered early after liftoff.

Analytical Models: Martin-Marietta Single-Node Pressurization Model. Model will need to be modified to simulate heat transfer from pressurization gases to LH₂ recirculation manifold after it is uncovered.

Test Requirements: Characterize any potential for ullage pressure to slump when recirc manifold is uncovered via analysis and then subscale, single engine testing. Final characterization, plus verification of new GH₂ pressurization system control setting, should be made as part of the Systems Integration Verification Test (see Item 22) during FRF.

Risk Assessment: GH₂ press system performance margin should be more than adequate to compensate for the possibility of a very slight ullage pressure slump when the cold recirc manifold is uncovered. Should the changes required to maintain the LH₂ ullage pressure at a lower control band not be completely verified, then the worst possible failures would be for all the flow control valves to remain either open or closed throughout the flight. Consequences of overpressurization would be tank venting during ascent which carries the potential for fire/explosion at lower altitudes. Consequences of under pressurization might be failure to meet engine NPSP requirements and possibly the collapse of the LH₂ tank.

STS Propellant Densification Test Requirement Analysis

Item No.: 8 **Requirement:** LO₂ Flight Pressurization Performance

Description: Changes that will affect LO₂ pressurization system performance are: (1) colder, denser LO₂, (2) SLWT, (3) LO₂ recirculation manifold, and (4) change back to an active GO₂ flow control system. Items (1), (2) and (3) will affect the ullage thermodynamics and tank dynamics and may significantly worsen the LO₂ ullage pressure slump experienced shortly after liftoff. See Item 21 for related test requirements.

Analytical Models: Martin-Marietta Single-Node Pressurization Model.

Test Requirements: Analytically determine the performance of an active GO₂ flow control system with colder LO₂ and a recirc manifold heat sink. Test for effects on ullage pressure when recirc manifold is uncovered during subscale, single engine testing. Final characterization, plus verification of the reactivated GO₂ pressurization system, should be made as part of the Systems Integration Verification Test (see Item 22) during FRF.

Risk Assessment: GO₂ press system performance margin should be more than adequate to compensate for the possibility of a very slight ullage pressure slump when the cold recirc manifold is uncovered. Should reactivation of an active GO₂ flow control system not be completely verified, then the worst possible failures would be for all the flow control valves to remain either open or closed throughout the flight. Consequences of overpressurization would be tank venting of GO₂ during ascent which carries some potential for fire/explosion if an ignition source is available. Consequences of under pressurization might be the collapse of the LO₂ tank.

STS Propellant Densification Test Requirement Analysis

Item No.: 9 **Requirement:** KSC LH₂ Procedures

Description: All nominal and contingency procedures must be tested for loading and draining densified LH₂. After the densification system is started following topping, procedures which are affected by the densified LH₂ are replenish, stop flows, reverts back to fast fill, topping and earlier stages of replenish, and drain.

Analytical Models: None required

Test Requirements: Obtain KSC support as early as possible to coordinate testing of effects densified LH₂ would have on loading and draining operations. During series of densification tests with subscale tanks, conduct (1) normal replenish and drain operations, (2) stop flows of varying duration, (3) reverts to various loading phases, and (4) hold time assessment after stop flows, replenish termination and pre-press. Maximum hold times for maintaining subcooled LH₂ should be determined. Transition sequencing back to full flow should be determined. During the KSC loading tests, many of these same subscale tests should be repeated for fullscale tank characterization.

Risk Assessment: Risks involved with insufficient characterization of loading and draining operations for densified LH₂ are (1) possible hardware damage during unsuitable flow transition, (2) countdown delays in establishing LH₂ flight mass, and (3) uncertainty in hold time available following stop flows and terminate replenish.

STS Propellant Densification Test Requirement Analysis

Item No.: 10 **Requirement:** KSC LO₂ Procedures

Description: All nominal and contingency procedures must be tested for loading and draining densified LO₂. After the densification system is started following topping, procedures which are affected by the densified LO₂ are replenish, stop flows, reverts back to fast fill, topping and earlier stages of replenish, and drain.

Analytical Models: None required

Test Requirements: Obtain KSC support as early as possible to coordinate testing of effects densified LO₂ would have on loading and draining operations. During series of densification tests with subscale tanks, conduct (1) normal replenish and drain operations, (2) stop flows of varying duration, (3) reverts to various loading phases, and (4) hold time assessment after stop flows, replenish termination and pre-press. Maximum hold times for maintaining subcooled LO₂ should be determined. Transition sequencing back to full flow should be determined. During the KSC loading tests, many of these same subscale tests should be repeated for fullscale tank characterization.

Risk Assessment: Risks involved with insufficient characterization of loading and draining operations for densified LO₂ are (1) possible hardware damage during unsuitable flow transition, (2) countdown delays in establishing LO₂ flight mass, and (3) uncertainty in hold time available following stop flows and terminate replenish.

STS Propellant Densification Test Requirement Analysis

Item No.: 11 **Requirement:** LH₂ Tank Stratification Characterization

Description: To maximize the LH₂ load during the densification process, good temperature mixing within the tank is required. Also, determination of the mean bulk temperature is required to accurately calculate the load on board. Propellant stratification should be investigated both analytically and via propellant tank temperature instrumentation.

Analytical Models: TBD

Test Requirements: Subscale LH₂ tank should be instrumented to determine densified LH₂ temperature profile horizontally and vertically. A major objective is to verify analytically modeling so that a flight tank will not require instrumentation. During densification tests, determine if, how and when proper propellant mixing is accomplished.

Risk Assessment: Lack of accurate temperature characterization would result in uncertainty in the mean bulk density and flight load.

STS Propellant Densification Test Requirement Analysis

Item No.: 12 **Requirement:** LO₂ Tank Stratification Characterization

Description: To maximize the LO₂ load during the densification process, good temperature mixing within the tank is required. Also, determination of the mean bulk temperature is required to accurately calculate the load on board. Propellant stratification should be investigated both analytically and via propellant tank temperature instrumentation.

Analytical Models: TBD

Test Requirements: Subscale LO₂ tank should be instrumented to determine densified LO₂ temperature profile horizontally and vertically. A major objective is to verify analytically modeling so that a flight tank will not require instrumentation. During densification tests, determine if, how and when proper propellant mixing is accomplished.

Risk Assessment: Lack of accurate temperature characterization would result in uncertainty in the mean bulk density and flight load.

STS Propellant Densification Test Requirement Analysis

Item No.: 13 **Requirement:** LH₂ Engine Recirculation Performance

Description: LH₂ recirculation performance via the clustered recirc system must be characterized for densified LH₂. Two objectives apply here: (1) avoid excessive propellant being lost through the hi-point bleed line following replenish termination, and (2) maintain SSME start temperature requirements.

Analytical Models: TBD

Test Requirements: Analytically assess engine LH₂ temperature conditioning during recirc pump operation with densified LH₂ and predict effect on hi-point bleed flow. Recirculation flow should be adjusted only if engine LH₂ temperatures are too cold, which is unlikely. Hi-point bleed flow can be reduced to lower the LH₂ drainback mass after replenish termination if adequate temperature margin is maintained (could be more trouble than it's worth). Verification of engine LH₂ temperature conditions should be made during the KSC loading test.

Risk Assessment: Typical LH₂ drainback mass is 104 lbm. With densified propellants, this may go up slightly and thus reduce the flight load, given the same hi-point bleed system configuration. Too cold LH₂ might increase the LH₂ oscillations during start.

STS Propellant Densification Test Requirement Analysis

Item No.: 14 **Requirement:** LO₂ Bleed Performance

Description: LO₂ bleed is required to eliminate heat entering the LO₂ feed system and engine. After replenish termination, the bleed goes through a drainback phase of nearly five minutes which reduces the LO₂ load on board by approximately 5,800 lbm. The ability to launch following an extended hold can be affected by the drainback time and rate. With colder LO₂, the bleed rate can probably be reduced to minimize the drainback mass while still meeting SSME temperature requirements.

Analytical Models: TBD

Test Requirements: Analytically assess SSME LO₂ conditioning for varying bleed rates. For densified LO₂, if the bleed rate can be reduced significantly and an adequate temperature margin still be maintained, then modify the engine bleed system and verify during single engine hot fire testing.

Risk Assessment: A bleed rate that is unnecessarily high sacrifices LO₂ flight mass during drainback. Also, a too high bleed rate (i.e., too cold LO₂ start conditions) may cause problems with the fuel preburner start and result in an oxygen rich start. A low nominal bleed rate could make the engine more sensitive to variations in the flow.

STS Propellant Densification Test Requirement Analysis

Item No.: 15 **Requirement:** LH₂ Feedline and Fill and Drain Line Performance

Description: Colder, denser LH₂ will affect the feedline pressure losses. These can be determined analytically. Also, valve operation should be verified with densified LH₂. Applicable valves are the 17 inch disconnect valves, the feedline prevalues, and the inboard and outboard fill and drain line valves.

Analytical Models: Develop spreadsheet simulation for feedline pressure losses.

Test Requirements: Determine pressure losses analytically. Determine valve performance with densified LH₂ through component test or by similarity with single engine test stand valving.

Risk Assessment: Pressure loss analysis is straight forward with no significant risk involved. Risk of not verifying proper valve operation under densified LH₂ conditions prior to KSC loading test would be to experience faulty valve operation while on the pad and thus delay schedule to accommodate further valve testing and possible changeout.

STS Propellant Densification Test Requirement Analysis

Item No.: 16 **Requirement:** LO₂ Feedline and Fill and Drain Line Performance

Description: Colder, denser LO₂ will affect the feedline pressure losses. These can be determined analytically. Also, valve operation should be verified with densified LO₂. Applicable valves are the 17 inch disconnect valves, the feedline prevalues, and the inboard and outboard fill and drain line valves.

Analytical Models: Develop spreadsheet simulation for feedline pressure losses.

Test Requirements: Determine pressure losses analytically. Determine valve performance with densified LO₂ through component test or by similarity with single engine test stand valving.

Risk Assessment: Pressure loss analysis is straight forward with no significant risk involved. Risk of not verifying proper valve operation under densified LO₂ conditions prior to KSC loading test would be to experience faulty valve operation while on the pad and thus delay schedule to accomodate further valve testing and possible changeout.

STS Propellant Densification Test Requirement Analysis

Item No.: 17 **Requirement:** POGO System Performance

Description: LO₂ POGO suppression is accomplished with an SSME mounted POGO accumulator. Any effect that densified LO₂ might have on accumulator performance should be determined analytically and verified through single engine testing with densified LO₂.

Analytical Models: TBD

Test Requirements: Analytically assess POGO accumulator performance with densified LO₂. Verify proper accumulator performance on single engine test stand. Proper performance would mean elimination of LO₂ feed system oscillations coupled with engine and maintaining proper accumulator He charge performance.

Risk Assessment: Worst case risk would be collapse or loss of accumulator He charge and subsequent coupling of engine oscillations with LO₂ feedlines.

STS Propellant Densification Test Requirement Analysis

Item No.: 18 **Requirement:** Nose Cone & Intertank Purge Performance Verification

Description: Colder vent lines plus recirculation lines placed in the nose cone and intertank may affect the performance of the heated GN₂ purges in these compartments. This may require an increase in the purge supply pressure and/or temperature.

Analytical Models: Martin-Marietta Intertank conditions model.

Test Requirements: Analytically determine if purge system supply pressure and temperature should be modified to maintain required nose cone and intertank conditions. Adjust purge system output prior to KSC loading test and verify during loading with compartment temperature and pressure data.

Risk Assessment: Intertank: Possible RSS battery and electronics problems and structural problem with colder temperatures. Nose cone: Possible icing problem around vent ducts.

STS Propellant Densification Test Requirement Analysis

Item No.: 19 **Requirement:** SSME Start/Shutdown Transient Performance

Description: Densified propellants will change the SSME inlet temperatures, pressures and densities and will affect SSME transient performance during start and shutdown. These changes may affect preburner performance and require changes in valve scheduling.

Analytical Models: SSME Digital Transient Model

Test Requirements: Analytical simulations are questionable for transient performance. Therefore, short duration single engine hot-fire tests should be conducted at SSC with progressively colder propellants. During start transients, evaluate OPB start pops and any difficulty in igniting all OPB injector elements due to possible reduction in fuel oscillation magnitude and due to the reduced LO₂ manifold boiloff. Adjust OPOV schedule as may be required. Evaluate FPB for increased fuel flow and reduced preburner LO₂ injector flow at ignition. Adjust FPOV schedule as may be required. During shutdown transients, evaluate possible increase in number of small preburner pops due to colder LO₂ residuals. This is for characterization testing, as there is no known corrective action.

Risk Assessment: Skipping the testing for start and shutdown performance is not an option. Improper testing (e.g., starting with fully densified propellants on first test) could result in such poor preburner performance that a damaging engine shutdown could result.

STS Propellant Densification Test Requirement Analysis

Item No.: 20 **Requirement:** SSME Mainstage Operation

Description: Densified propellants will change the SSME inlet temperatures, pressures and densities and will affect SSME mainstage performance. These changes may affect mixture ratio, turbine temps and nozzle cooling margins.

Analytical Models: SSME Power Balance Model

Test Requirements: Conduct single engine hot-fire tests at SSC. During mainstage, evaluate for mixture ratio effects and reduced turbine temps and increased MCC/nozzle cooling margins. This testing is to characterize engine performance rather than to adjust for changes.

Risk Assessment: Improper engine mainstage characterization would result in inaccurate redlines, and engine performance errors (i.e., specific impulse, thrust, propellant consumption, etc.) that could effect vehicle velocities and consume propellant reserves.

STS Propellant Densification Test Requirement Analysis

Item No.: 21 **Requirement:** LO₂ Ullage Pressure Slump Verification

Description: The LO₂ ullage pressure slump that occurs shortly after liftoff is a result primarily of a small ullage and propellant splash caused by the SRM thrust buildup that sets off a tank "breathing motion." Colder LO₂ will contribute to the slump. The LO₂ recirc manifold will act as a heat sink after it is uncovered and may also contribute to the slump. The breathing motion of the SLWT may be different from the ET LWT. All of these things need to be assessed because this is a very critical phenomenon. An ullage pressure slump larger than what the tank is capable of withstanding could result in the loss of the vehicle and crew.

Also, a precise determination of the SLWT LO₂ low pressure structural capability is important. This would provide the most accurate margin between what the tank could withstand and the predicted minimum pressure during the period of slump.

Analytical Models: Martin-Marietta Single-Node Pressurization Model. CFD Research, Incorporated's LO₂ Ullage Pressure Slump CFD Model.

Test Requirements: This is a very difficult area to test for but may be extremely important. Of course the dynamics of SRB thrust buildup on the LO₂ tank cannot be tested directly. However, dynamics tests of a subscale tank that could closely duplicate the predicted breathing motion of the ET LO₂ tank (when loaded to various levels with densified LO₂) and thus produce the correct "splash" at the LO₂ surface would be required. The resultant ullage pressure slump could then be evaluated.

Risk Assessment: Uncertainties in analytically modeling of the LO₂ ullage pressure slump, coupled with the potentially disastrous results of a slump that would violate LO₂ tank minimum structural capabilities, make it imperative to minimize uncertainties in this area. If this cannot be done, then the first flight with densified propellants might require a larger than normal ullage (i.e., lowered LO₂ load) to reduce the slump potential. This could change mission objectives and affect the Space Station schedule.

STS Propellant Densification Test Requirement Analysis

Item No.: 22 **Requirement:** Systems Integration Verification

Description: The densified propellant change would likely be incorporated with several other significant shuttle changes: SLWT, Block II SSME's, various orbiter mods, and a return to active GOX flow control system. With the number and nature of the changes, it would be prudent to conduct a short duration Flight Readiness Firing (FRF) on the launch pad. An FRF would verify proper systems integration, and identify correlated performance effects, if any, that would not show up under individual system testing. Also, this would be the only opportunity to test for cluster effects and changes to the vehicle twang.

Analytical Models: Shuttle Integrated Performance Prediction Simulation (SIPPS).

Test Requirements: Conduct normal propellant loading and count down followed by a short duration (20 seconds should suffice) hot-firing. Place priority on evaluating twang, LO₂ pressurization system performance, LH₂ pressurization system performance, and engine inlet conditions and engine control with densified propellants and clustered performance.

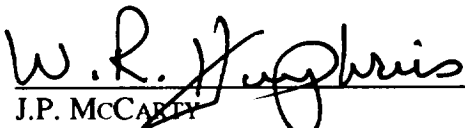
Risk Assessment: Risks are difficult to predict. Uncertainties in integration of all systems, especially systems that have been changed, and correlated performance effects that we are not smart enough to identify or cannot test for with systems tests are the primary risks. Problems could conceivably scrub a mission or actually sacrifice a mission if not discovered prior to an actual launch attempt.

APPROVAL

STS PROPELLANT DENSIFICATION FEASIBILITY STUDY DATA BOOK

By M.M. Fazah

The information in this report has been reviewed for technical content. Review of any information concerning Department of Defense or nuclear energy activities or programs has been made by the MSFC Security Classification Officer. This report, in its entirety, has been determined to be unclassified.



J.P. McCARTY
Director, Propulsion Laboratory

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13. ABSTRACT (Maximum 200 words) The feasibility of using densification or subcooling with respect to standard temperature propellants on the Space Transportation System (STS) in order to achieve a payload gain is discussed in this report. The objective is to determine the magnitude of the payload gain and to identify any system impacts to the space shuttle on either flight systems or ground systems. Results show that a payload benefit can be obtained by subcooling the liquid hydrogen (LH ₂) from a nominal temperature of 36.4 °R to 28.5 °R and by subcooling the liquid oxygen (LO ₂) from a nominal temperature of 164 °R to either 132.1 °R or 141.4 °R. When the propellants are subcooled to 28.5 °R and 132.1 °R for the LH ₂ and LO ₂ , respectively, a maximum payload gain of 7,324 lb can be achieved, and when the propellants are subcooled to 28.5 °R and 141.5 °R for the LH ₂ and LO ₂ , respectively, a maximum payload gain of 6,841 lb can be achieved. If the LH ₂ is subcooled to 28.5 °R while the LH ₂ and LO ₂ remains at the nominal conditions, a maximum payload gain of 1,303 lb can be achieved.				
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